



(19)

Europäisches Patentamt  
European Patent Office  
Office européen des brevets



(11)

EP 1 054 396 A3

(12)

## EUROPEAN PATENT APPLICATION

(88) Date of publication A3:  
**06.12.2000 Bulletin 2000/49**

(51) Int Cl.7: **G11B 15/67**

(43) Date of publication A2:  
**22.11.2000 Bulletin 2000/47**

(21) Application number: **00304193.6**

(22) Date of filing: **18.05.2000**

(84) Designated Contracting States:  
**AT BE CH CY DE DK ES FI FR GB GR IE IT LI LU  
MC NL PT SE**

Designated Extension States:  
**AL LT LV MK RO SI**

(30) Priority: **18.05.1999 JP 13702899**

(71) Applicant: **NEC CORPORATION  
Tokyo (JP)**

(72) Inventor: **Wada, Satoshi, c/o NEC Yonezawa, Ltd.  
Yonezawa-shi, Yamagata (JP)**

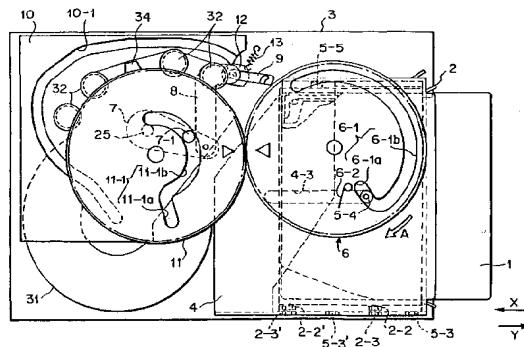
(74) Representative: **Orchard, Oliver John  
JOHN ORCHARD & CO.  
Staple Inn Buildings North  
High Holborn  
London WC1V 7PZ (GB)**

### (54) Tape apparatus mechanism

(57) A loading mechanism moves a cartridge tray (2) for accommodating a cartridge (1) between an eject position at which the cartridge (1) is mountable/demountable into/from the cartridge tray (2) and a mount position at which a reel of the cartridge is engaged with cartridge reel rotating means of the magnetic tape apparatus, wherein the cartridge tray is engaged with a loader drive plate having a loader drive roller (5-4) engaged with a loader drive cam groove (6-1) containing radial and circumferential direction portions (6-1a, 6-1b) in a loader drive gear (6). A threading mechanism moves a leader block (9) engageable with a leader pin affixed to the tip portion of a magnetic tape accommodated in the cartridge between an unload position in the neighbourhood of the cartridge tray (2) located at the mount position and a load position in a notch of the core portion of the reel (31) of the magnetic tape apparatus to which the leader block (9) is fitted, wherein the leader block (9) is engaged with an arm member (7, 8) having a threader drive roller (11-1) engaged with a threader drive cam groove (11-1) containing radial and circumferential direction portions (11-1a, 11-1b) formed in a threader drive gear (11) engaged with the loader drive gear (6). At the unload position, the leader block (9) is located at a retracted position at which the leader block is retracted from the cartridge (1) in the cartridge tray (2) located at the mount position, and a leader pin captured position at which the leader block (9) is engaged with the leader pin affixed to the tip portion of the magnetic tape in the cartridge. The loading mechanism and the threading mechanism are driven by a single driving

force generating source, wherein, as the loader drive gear (6) is rotated, the cartridge tray (2) is moved between the eject and mount positions, the leader block (9) is rotated by 90 degrees between the retracted and the leader pin captured positions and moved between the unload and load positions by means of a threading cam groove (10-1).

FIG.1A





European Patent  
Office

## EUROPEAN SEARCH REPORT

Application Number  
EP 00 30 4193

DOCUMENTS CONSIDERED TO BE RELEVANT									
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.)						
X	US 4 991 037 A (SHIMIZU MUNETAKA ET AL) 5 February 1991 (1991-02-05) * abstract; figures * * column 5, line 27 - column 11, line 47 * ---	1-24	G11B15/67						
E	WO 00 30090 A (KONINKL PHILIPS ELECTRONICS NV) 25 May 2000 (2000-05-25) * abstract; claims 1-4; figures * ---	1-24							
E	WO 00 30091 A (KONINKL PHILIPS ELECTRONICS NV) 25 May 2000 (2000-05-25) * page 10, line 24 - page 15, line 8; figures * ---	1-24							
E	WO 00 30098 A (KONINKL PHILIPS ELECTRONICS NV) 25 May 2000 (2000-05-25) * page 7, line 28 - page 12, line 13 * * abstract; figures * ---	1-24							
A	EP 0 368 667 A (NAKAMICHI CORP) 16 May 1990 (1990-05-16) * abstract; figures * ---	1,5,17, 18	TECHNICAL FIELDS SEARCHED (Int.Cl.)						
A	EP 0 339 148 A (NAKAMICHI CORP) 2 November 1989 (1989-11-02) * abstract; figures * ---	1,5,17, 18	G11B						
A	EP 0 326 369 A (NAKAMICHI CORP) 2 August 1989 (1989-08-02) * abstract; figures * ---	1,5,17, 18							
A	EP 0 293 267 A (NAKAMICHI CORP) 30 November 1988 (1988-11-30) * abstract; figures * -----	1,5,17, 18							
<p>The present search report has been drawn up for all claims</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="width: 33%;">Place of search</td> <td style="width: 33%;">Date of completion of the search</td> <td style="width: 33%;">Examiner</td> </tr> <tr> <td>THE HAGUE</td> <td>18 October 2000</td> <td>Declar, M</td> </tr> </table>				Place of search	Date of completion of the search	Examiner	THE HAGUE	18 October 2000	Declar, M
Place of search	Date of completion of the search	Examiner							
THE HAGUE	18 October 2000	Declar, M							
CATEGORY OF CITED DOCUMENTS		T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document							
X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document									

ANNEX TO THE EUROPEAN SEARCH REPORT  
ON EUROPEAN PATENT APPLICATION NO.

EP 00 30 4193

This annex lists the patent family members relating to the patent documents cited in the above-mentioned European search report.  
The members are as contained in the European Patent Office EDP file on  
The European Patent Office is in no way liable for these particulars which are merely given for the purpose of information.

18-10-2000

Patent document cited in search report		Publication date		Patent family member(s)	Publication date
US 4991037	A	05-02-1991		JP 62223848 A JP 62226456 A JP 62226463 A JP 62226464 A JP 62226465 A JP 62226466 A	01-10-1987 05-10-1987 05-10-1987 05-10-1987 05-10-1987 05-10-1987
WO 0030090	A	25-05-2000		NONE	
WO 0030091	A	25-05-2000		NONE	
WO 0030098	A	25-05-2000		NONE	
EP 0368667	A	16-05-1990	US	5046168 A	03-09-1991
EP 0339148	A	02-11-1989	JP	1780039 C JP 4072301 B JP 63108551 A US 4832284 A	13-08-1993 17-11-1992 13-05-1988 23-05-1989
EP 0326369	A	02-08-1989	JP	1192049 A	02-08-1989
EP 0293267	A	30-11-1988	US	4879614 A	07-11-1989



**EP 1 054 396 B1**

(12)

## EUROPEAN PATENT SPECIFICATION

(45) Date of publication and mention  
of the grant of the patent:  
**13.06.2007 Bulletin 2007/24**

(51) Int Cl.:  
**G11B 15/67 (2006.01)**

(21) Application number: **00304193.6**

(22) Date of filing: **18.05.2000**

### (54) Tape apparatus mechanism

Mechanismus für Bandgerät

Mécanisme d'appareil à bande

(84) Designated Contracting States:  
**FR GB**

(30) Priority: **18.05.1999 JP 13702899**

(43) Date of publication of application:  
**22.11.2000 Bulletin 2000/47**

(73) Proprietor: **NEC CORPORATION**  
**Tokyo (JP)**

(72) Inventor: **Wada, Satoshi**,  
**c/o NEC Yonezawa, Ltd.**  
**Yonezawa-shi**,  
**Yamagata (JP)**

(74) Representative: **W.P. Thompson & Co.**  
**55 Drury Lane**  
**London WC2B 5SQ (GB)**

(56) References cited:

<b>EP-A- 0 293 267</b>	<b>EP-A- 0 326 369</b>
<b>EP-A- 0 339 148</b>	<b>EP-A- 0 368 667</b>
<b>WO-A-00/30090</b>	<b>WO-A-00/30091</b>
<b>WO-A-00/30098</b>	<b>US-A- 4 991 037</b>

**EP 1 054 396 B1**

Note: Within nine months from the publication of the mention of the grant of the European patent, any person may give notice to the European Patent Office of opposition to the European patent granted. Notice of opposition shall be filed in a written reasoned statement. It shall not be deemed to have been filed until the opposition fee has been paid. (Art. 99(1) European Patent Convention).

## Description

**[0001]** The present invention relates to a technique for magnetic recording and reproduction. A particular loading and threading mechanism of a magnetic tape apparatus using a single reel cartridge magnetic tape as a storage medium will be described below, by way of example, in illustration of the present invention.

**[0002]** A single reel cartridge magnetic tape is used as one of the storage media for electronic computers. The magnetic tape is secured to one reel at one end portion thereof and wound around the reel. The other end portion of the magnetic tape is provided with a leader block, and the magnetic tape is accommodated within a cartridge while the end portion having the leader block is on the outside.

**[0003]** When the cartridge is inserted into a magnetic tape apparatus, the cartridge is first fed to and mounted at a predetermined position of the magnetic tape apparatus (loading operation), and then the leader block of the tip portion of the cartridge is moved through a predetermined route, whereby the magnetic tape is fitted to a reel at the side of the magnetic tape apparatus (machine reel) while the magnetic tape is arranged in a predetermined running route (threading operation).

**[0004]** As disclosed in JP-8-17111(A) or JP-9-128855 (A), for example, the loading operation and the threading operation in a single reel cartridge magnetic tape apparatus have been hitherto performed by using separate mechanisms. That is, a loading mechanism for mounting the cartridge in the magnetic tape apparatus, and a tape threading mechanism for feeding the tip portion of the magnetic tape of the cartridge to the reel of the magnetic tape apparatus are driven by respective sources generating driving forces which are independent of each other.

**[0005]** Thus, in the loading and threading mechanisms of previously proposed cartridge magnetic tape apparatus, the number of driving force generating sources is comparatively large and thus the driving circuits for the respective driving force generating sources must be provided separately. In addition, in order to make properly the connection between the operation of the loading mechanism and the operation of the threading operation, a number of sensors for detecting the operating states of the loading mechanism and the threading mechanism must be provided for the magnetic tape apparatus, both of the mechanisms being controlled on the basis of signals from these sensors. Therefore, the control circuit is complicated, and it makes it difficult to miniaturize the apparatus and reduce the cost.

**[0006]** US4991037, which comprises the features of the preamble of claim 1, discloses a tape drive for withdrawing a magnetic tape together with a leader block connected to the leading end thereof out of a tape cartridge so as to record and reproduce the information onto or off of said magnetic tape.

**[0007]** WO00/30090 (priority date 12.11.1998; publication date 25.05.2000) discloses a recording device

which comprises movable holder means for holding a cassette which includes a record carrier in the form of a tape and a coupling element connected to the free end of the record carrier, the holder means being movable

5 from a loading position into an operating position in a motor-driven manner, and which comprises retaining means movable between a standby position and an operating position, for retaining a pull-out element which can be coupled to the coupling element to form a pull-out assembly when the holder means including a cassette have been moved into their operating position.

**[0008]** WO00/30091 (priority date 12.11.1998 ; publication date 25.05.2000) discloses a storage device and a storage container which can be loaded into the storage device, the storage device having holder means for holding a storage container which are guided so as to be movable between a loading position and an operating position and which are movable out their loading position in a first direction of movement and which are movable 15 into their operating position in a second direction of movement which oriented transversely to the first direction of movement.

**[0009]** WO00/30098 (priority date 12.11.1998; publication date 25.05.2000) discloses a recording device 20 similar to that of WO00/30090.

**[0010]** Features of arrangements to be described below, by way of example in illustration of the present invention, are that the number of sources generating the driving forces are reduced, and the number of operating 25 state detecting sensors required for the driving control are reduced, thereby enabling a control circuit in a mechanism for loading and threading of a cartridge magnetic tape apparatus, to be simplified, with the aim of miniaturizing the apparatus and reducing the cost.

**[0011]** According to an aspect of the present invention, there is provided a loading/threading mechanism for a single reel cartridge magnetic tape apparatus, including a loading mechanism for moving a cartridge tray for accommodating a cartridge between an eject position in 30 which the cartridge is mountable/ demountable into/from the cartridge tray and a mount position in which a reel of the cartridge is engaged with a cartridge reel rotating means of the magnetic tape apparatus; and a threading mechanism for moving the tip portion of a magnetic tape 35 accommodated in the cartridge between a position in the cartridge tray located at the mount position and a position in a reel of the magnetic tape apparatus, the loading mechanism and the threading mechanism being driven by means of a single driving force generating source; 40 characterized in that:

50 the threading mechanism includes a leader block which is detachably engageable with a leader pin affixed to the tip portion of the magnetic tape; the leader block is moved through a predetermined route between an unload position in the neighbourhood of the cartridge tray located at the mount position and a load position in a notch of the core portion

of the reel of the magnetic tape apparatus; and at the unload position, the leader block is located at a retracted position at which the leader block is disengaged from the leader pin and retracted from the cartridge accommodated in the cartridge tray located at the mount position, and a leader pin captured position at which the leader block is engaged with the leader pin affixed to the tip portion of the magnetic tape in the cartridge accommodated in the cartridge tray located at the mount position.

**[0012]** Preferably, the loading mechanism has a loader drive rotator with a loader drive cam groove, and a loader drive member which is reciprocally movable in one direction to translate the cartridge tray between the eject position and the mount position, the loader drive member having a loader drive engaging member which is engaged with the loader drive cam groove, the threading mechanism having a threader drive rotator with a threader drive cam groove and a threader drive member for driving the leader block to move between the unload position and the load position, the threader drive member having a threader drive engaging member for engagement with the threader drive cam groove, the loader drive rotator and the threader drive rotator being connected to each other so that a rotational driving force is transmitted therebetween, and a driving force is transmitted from the driving force generating source to one of the loader drive rotator and the threader drive rotator.

**[0013]** Conveniently, the loading mechanism has a loader drive cam groove formed in a driving rotator, and a loader drive member which is reciprocally movable in one direction to translate the cartridge tray between the eject position and the mount position, the loader drive member having a loader drive engaging member for engagement with the loader drive cam groove, the threading mechanism having a threader drive cam groove in the driving rotator, and a threader drive member for driving the leader block to move between the unload position and the load position, the threader drive member having a threader drive engaging member for engagement with the threader drive cam groove, whereby a driving force is transmitted from the driving force generating source to the driving rotator.

**[0014]** Further, the threading mechanism has a retractor for moving the loader drive member in the condition in which the cartridge tray is disposed at the mount position, thereby shifting the leader block from the retracted position to a leader pin captured position, and the retractor having a pin which is engageable with a retractor cam secured to the loader drive member, and a groove which is engageable with a pin secured to the leader block.

**[0015]** Further, the loading mechanism and the threading mechanism include a loader drive cam groove and a threader drive cam groove formed on a rotating member, respectively, the loader drive cam groove having a first groove portion extending in the circumferential direction so as to be kept away from the rotational centre of the

rotating member by a fixed distance, and a second groove portion which is linked to one end portion of the first groove portion and extends so that the distance from the rotational centre of the rotating member is varied, the

5 loader drive cam groove being engaged with a loader drive engaging member affixed to a loader drive member, the loader drive member being engaged with the cartridge tray, whereby in the condition in which the second groove portion is engaged with the loader drive engaging member, during the rotation of the rotating member, the cartridge tray is moved between the eject position and the mount position, the threader drive cam groove having a third groove portion extending in the circumferential direction so as to be kept away from the rotational centre

10 of the rotating member by a fixed distance, and a fourth groove portion which is linked to one end portion of the third groove portion and extends so that the distance from the rotational centre of

15 the rotating member is varied, the threader drive cam groove being engaged with a threader drive engaging member affixed to a threader drive member, the threader drive member being connected to the leader block, whereby in the condition in which the fourth groove portion is engaged with the threader drive engagement member, during the rotation of the rotating member the leader block is moved through the predetermined route, the first to fourth groove portions being set so that when the loader drive engaging member is engaged with the second groove portion, the threader drive engaging member is engaged with the third groove portion, and when the threader drive engaging member is engaged with the fourth groove portion, the loader drive engaging member

20 25 30 35

the rotating member is varied, the threader drive cam groove being engaged with a threader drive engaging member affixed to a threader drive member, the threader drive member being connected to the leader block, whereby in the condition in which the fourth groove portion is engaged with the threader drive engagement member, during the rotation of the rotating member the leader block is moved through the predetermined route, the first to fourth groove portions being set so that when the loader drive engaging member is engaged with the second groove portion, the threader drive engaging member is engaged with the third groove portion, and when the threader drive engaging member is engaged with the fourth groove portion, the loader drive engaging member is engaged with the first groove portion, and the single driving force generating source transferring the rotational force to the rotating member.

**[0016]** Also, the rotating member includes a first rotator 40 and a second rotator which are connected to each other so as to be rotatable in synchronism with each other, the loader drive cam groove being in the first rotator, and the threader drive cam groove being in the second rotator. In another arrangement illustrative of the present invention,

45 the first rotator is a loader drive gear, and the second rotator is a threader drive gear, the loader drive gear and the threader drive gear being engaged with each other.

**[0017]** Preferably, the loader drive member moves the cartridge tray while a guide engaging member affixed to the cartridge tray is guided in engagement with a loader guide groove formed in a loader guide member. A feature of one arrangement illustrative of the present invention, is that the loader guide groove is bent.

**[0018]** Conveniently, the leader block is located at one 50 of the retracted position and the leader pin captured position in the unload position.

**[0019]** Further, a retractor block is provided and is rotatable to rotate the leader block located at the unload

position, the retractor block being biased in one rotational direction, the loader drive member being provided with a retractor cam which is engaged with the retractor block to rotate the retractor block in the other rotational direction, and the loader drive member being moved while the cartridge tray is located at the mount position, whereby the leader block is rotated against the biasing force to be shifted from the retracted position to the leader pin captured position. In a particular illustrative arrangement there is a guide cam for restricting the rotational angle range of the retractor block in the unload position.

**[0020]** Also, the cartridge tray has an engaging member for engagement with a door of the cartridge inserted to the eject position to open the door. The cartridge tray may be provided with a lock member which is engaged with a lock hole of the cartridge inserted to the eject position to fix the cartridge to the cartridge tray. There may be included a lock mechanism for fixing the cartridge tray to the eject position, the lock mechanism abutting against the slanting surface of the cartridge to release the fixing. The leader block may be rotatably held with a clearance by a support pin secured to the threader drive member, and a threading cam groove is arranged for guiding the support pin when the leader block is moved between the unload position and the load position.

**[0021]** Preferably, the loading mechanism includes a loading mechanism portion and the threading mechanism includes a threading mechanism portion, in which the loading mechanism portion has a loader drive gear, and a loader drive cam groove, including a circumferential direction groove portion and a non-circumferential direction groove portion in the loader drive gear, the loader drive cam groove being engaged with a loader drive engaging member affixed to a loader drive plate, the loader drive plate being engaged with the cartridge tray, whereby when the non-circumferential direction groove portion of the loader drive cam groove is engaged with the loader drive engaging member, during the rotation of the loader drive gear, the cartridge tray is moved between the eject position and the mount position and in which, the threading mechanism portion has a threader drive gear engageable with the loader drive gear, and a threader drive cam groove, including a circumferential direction groove portion and a non-circumferential direction groove portion is formed in the threader drive gear, the threader drive cam groove being engaged with a threader drive engaging member which is affixed to a threader drive arm member rotatable around the rotating centre parallel to the rotational centre of the threader drive gear so as to be eccentric to the rotating centre, and the threader drive arm member is connected to the leader block, whereby when the non-circumferential direction groove portion of the threader drive cam groove is engaged with the threader drive engaging member, during the rotation of the threader drive gear, the leader block is moved through the predetermined route between the unload position and the load position, the loader drive gear and the threader drive gear being engaged with each other, so that when the

loader drive engaging member is engaged with the non-circumferential direction groove portion of the loader drive cam groove, the threader drive engaging member is engaged with the circumferential direction groove portion of the threader drive cam groove, and when the threader drive engaging member is engaged with the non-circumferential direction groove portion of the threader drive cam groove, the loader drive engaging member is engaged with the circumferential direction

- 5 groove portion of the loader drive cam groove, and the single driving force generating source is a driving motor for driving the rotation of the loader drive gear or the threader drive gear.
- 10 **[0022]** Further, the loading mechanism includes a loader drive cam groove in a driving rotator and the threading mechanism includes a threader drive cam groove in the driving rotator, the loader drive cam groove including a first circumferential direction groove portion and a first non-circumferential direction groove portion
- 15 20 and the threader drive cam groove having a second circumferential direction groove portion and a second non-circumferential direction groove portion, the loader drive cam groove being engaged with a loader drive engaging member affixed to a loader drive plate, the loader drive
- 25 plate being engaged with the cartridge tray, whereby when the first non-circumferential direction groove portion of the loader drive cam groove is engaged with the loader drive engaging member, during the rotation of the rotator, the cartridge tray is moved between the eject
- 30 35 position and the mount position, the threader drive cam groove being engaged with a threader drive engaging member which is affixed to a threader drive arm member rotatable around the rotating centre parallel to the rotational centre of the rotator so as to be eccentric to the rotating centre, the threader drive arm being connected to the leader block, whereby when the second non-circumferential direction groove portion of the threader drive cam groove is engaged with the threader drive engaging member, during the rotation of the rotator, the leader
- 40 45 block is moved through the predetermined route between the unload position and the load position, and the loader drive cam groove and the threader drive cam groove being set to have such a phase that when the loader drive engaging member is engaged with the first non-circumferential direction groove portion of the loader drive cam groove, the threader drive engaging member is engaged with the second circumferential direction groove portion of the threader drive cam groove, and when the threader drive engaging member is engaged with the second non-circumferential direction groove portion of the threader drive cam groove, the loader drive engaging member is engaged with the first circumferential direction groove portion of the loader drive cam groove.
- 50 **[0023]** Also, there is provided a retractor block engageable with the leader block located at the unload position, wherein the retractor block is rotatable between a first attitude and a second attitude and biased so as to assume the first attitude as a result of the biasing means,

and the retractor block is engaged with the leader block, so that when in the first attitude the leader block is retracted from the cartridge accommodated in the cartridge tray located at a mount position, while in the second attitude the leader block is advanced to the cartridge accommodated in the cartridge tray located at the mount position, the retractor block being set to the second attitude against the biasing force of the biasing means when a retractor cam affixed to the loader drive plate abuts against the retractor block. In one illustrative arrangement, the biasing means includes a spring. In another illustrative arrangement, the leader block is rotatably secured to a threader drive arm member and has an engaging projection which is engageable with the retractor block, and the retractor block has an engaging groove which is engageable with the engaging projection.

**[0024]** Preferably, a leader pin capture groove, which is engageable with the leader pin, is formed in the leader block. A threading guide groove for setting the predetermined route of the movement of the leader block may be provided, and the threader drive arm member has a threading guide engaging member which is engageable with the threading guide groove. The threader drive arm member may include a first arm portion which is rotatable around the rotating centre and which has the threader drive engaging member, and a second arm portion which is rotatably connected to the first arm portion and connected to the leader block.

**[0025]** In a loading/threading mechanism of one arrangement illustrative of the present invention, the movement of a cartridge tray between an eject position and a mount position, the movement of a leader block at the unload position between a retract position and a leader pin capture position and the movement of the leader block between the leader pin capture position and the load position can be performed by means of sequential operations using a single source for generating a driving force. Accordingly, the state of the loading/threading operation can easily and simply be detected, and the number of detecting sensors for detecting the state of the mechanism and for the control of the apparatus can be kept to a minimum. Further, the number of sources for generating a driving force can be reduced. Therefore, the control circuit of the apparatus can be simplified and the apparatus can be miniaturized in size and reduced in cost.

**[0026]** Further, the previously proposed loading/threading mechanism needs a lock mechanism in order to inhibit the ejecting operation of the cartridge when the leading end of the magnetic tape is out of the cartridge. In the arrangements to be described below, by way of example in illustration of the present invention, no lock mechanism is required, because a sequential operation is carried out by a single source generating a driving force.

**[0027]** The following description and drawings disclose, by means of examples, the invention which is defined in the appended claims, whose terms determine the extent of the protection conferred hereby.

**[0028]** In the drawings:-

5 Fig. 1A is a plan view of a loading/threading mechanism, and Fig. 1 B is a partial, enlarged view of Fig. 1 A,  
Fig. 2 is a side view of the loading/threading mechanism of Figs. 1A and 1B,  
Fig. 3 is a perspective view showing a single reel cartridge,  
10 Fig. 4 is a partial side view showing the single reel cartridge,  
Fig. 5 is a partial, sectional view showing the single reel cartridge,  
Fig. 6 is a front view of the loading/threading mechanism of Figs. 1A and 1B,  
Fig. 7 is a partial, exploded, perspective view of a loading mechanism portion of the loading/threading mechanism of Figs. 1A and 1B,  
Fig. 8 is a partial, sectional view showing a connection portion between a leader block and a threading arm in the loading/threading mechanism of Figs. 1 A and 1 B,  
20 Fig. 9 is a plan view showing the leader block of the loading/threading mechanism of Figs. 1 A and 1 B,  
Fig. 10 is an exploded, perspective view showing the engagement relationship between the leader block, a retractor block and the peripheral portions thereof in the loading/threading mechanism of Figs. 1 A and 1 B,  
25 Fig. 11 is a plan view showing the engagement relationship between a machine reel and the leader block in the loading/threading mechanism of Figs. 1A and 1B,  
Figs. 12 A and 12B are cross-sectional views showing a cartridge lock mechanism to a cartridge tray in the loading/threading mechanism of Figs. 1 A and 1 B,  
30 Figs. 13A and 13B are cross-sectional views showing a cartridge tray lock mechanism of the loading/threading mechanism of Figs. 1 A and 1 B,  
Fig. 14 is a plan view showing the loading/threading mechanism of Figs. 1 A and 1B,  
35 Fig 15 is a side view of the loading/threading mechanism of Figs. 1A and 1 B,  
Fig. 16 is a plan view the loading/threading mechanism of Figs. 1A and 1 B,  
Fig. 17 is a side view of the loading/threading mechanism of Figs. 1A and 1 B,  
40 Fig. 18 is a partial, cross-sectional view showing a leader pin capture state in the loading/threading mechanism of Figs. 1 A and 1B,  
Figs 19A and 19B are plan views showing a guide cam in the loading/threading mechanism of Figs. 1 A and 1B,  
45 Fig. 20 is a plan view of the loading/threading mechanism of Figs. 1A and 1B,  
Fig. 21 is a plan view of the loading/threading mechanism of Figs. 1A and 1B,

50

55

Figs. 22A and 22B are plan views showing the engagement relationship between the machine reel and the leader block in the loading/threading mechanism of Figs. 1A and 1B, and Fig. 23 is a plan view of another loading/threading mechanism.

**[0029]** Fig. 1A is a plan view showing the overall structure of a loading/threading mechanism of a cartridge magnetic tape apparatus, and Fig. 1 B is a partial, enlarged view of Fig. 1 A. Fig. 2 is a side view of the loading/threading mechanism. In the mechanism, a loading mechanism portion and a threading mechanism portion are arranged in a predetermined connecting relationship on a deck base 3. In Fig. 1A, the loading mechanism portion is mainly located at the right side and the threading mechanism portion is mainly located at the left side. **[0030]** The loading mechanism portion functions to feed, in the direction of an arrow X, a single reel cartridge 1 inserted in the direction of the arrow X to move the single reel cartridge 1 to a predetermined position at which a magnetic tape wound in the cartridge 1 can be withdrawn and rewound, and also feed the cartridge 1 having the magnetic tape rewound therein from the above predetermined position in the direction of an arrow Y to a position at which the cartridge 1 can be discharged in the direction of the arrow Y. The threading mechanism portion functions to draw out the tip or leading end portion of the magnetic tape from the cartridge 1 located at the predetermined position by the loading mechanism, to guide it to a take-up reel in the magnetic tape apparatus through a predetermined route passing by a magnetic head by which recording/reproducing can be carried out on the magnetic tape, and to connect the leading end of the magnetic tape to the core portion of the machine reel. When the magnetic tape is rewound into the cartridge 1, the threading mechanism portion also functions to pick up the tip or leading end portion of the magnetic tape from the core portion of the machine reel and move it back through the predetermined route to return it into the cartridge 1.

**[0031]** First, the single reel cartridge 1 will be described with reference to Figs. 3 to 5. Fig. 3 is a perspective view showing the cartridge 1. The cartridge 1 includes a casing 1-1, and a cartridge reel 1-2 which is accommodated in the casing 1-1 and around which a magnetic tape is wound. A door 1-3 is formed on the side end surface of the casing 1-1. Fig. 4 is a partial side view showing the cartridge 1 when the door 1-3 is opened, and Fig. 5 is a partial, cross-sectional view of the cartridge 1 of Fig. 4. **[0032]** The cartridge 1 is inserted into the loading mechanism portion in the direction of the arrow X shown in Figs. 3 and 4. The door 1-3 is biased or urged in the direction of the arrow X by biasing means or urging means such as a spring or the like (not shown), and the opening in the cartridge 1 is closed by the door 1-3 as shown in Fig. 3 before the cartridge 1 is inserted into the loading mechanism portion. An engaging projection 1-4

is provided on the door 1-3, and it is engaged with an engaging member in the loading mechanism portion when the cartridge 1 is loaded, whereby when the loading operation is completed, the door 1-3 is moved in the direction of the arrow Y to keep the cartridge open, as shown in Fig. 4.

**[0033]** In Fig. 3, only the lower surface side portion of the core portion of the cartridge reel 1-2 appears, together with a side-face gear 20. When the loading operation is completed, the side-face gear 20 of the reel engages a side-face gear 21 connected to a cartridge reel driving motor to enable a rotational drive to be provided to the cartridge reel 1-2. The cartridge reel 1-2 has a lock pin 1-5 projecting through the side-face gear thereof, and the lock pin 1-5 is downwardly biased until the cartridge reel 1-2 is engaged with the side-face gear 21 of a driving motor, whereby the rotation of the cartridge reel 1-2 relative to the casing 1-1 is prevented by a lock mechanism (not shown). The upward movement of the lock pin 1-5 through its engagement with the side-face gear 21 of the driving motor releases the lock to allow the rotation of the cartridge reel 1-2 relative to the casing 1-1.

**[0034]** As shown in Figs. 4 and 5, a leader pin 1-7 is secured to the tip of the magnetic tape 1-6 wound around the cartridge reel 1-2. The leader pin 1-7 is accommodated in a predetermined accommodating portion 1-8 in the casing 1-1 as shown in Figs. 4 and 5 in the condition in which the magnetic tape 1-6 is wound around the cartridge reel 1-2, whereby the leader pin 1-1 is positioned with respect to the casing 1-1.

**[0035]** In the loading mechanism portion, a loader guide plate 4 is arranged so as to be fixed in position with respect to the deck base 3, a loader drive plate 5 is arranged inside the loader guide plate 4, and a cartridge tray 2 is arranged inside the loader drive plate 5, as shown in Figs. 1 and 2. Fig. 6 is a front view of this embodiment, which is viewed in the cartridge insertion direction. Fig. 7 is a partially exploded, perspective view showing the relationships between the loader guide plate 4, the loader drive plate 5 and the cartridge tray 2.

**[0036]** As shown in Fig. 7, the cartridge 1 is inserted into the cartridge tray 2 in the direction of the arrow X. A pin 2-1 is arranged on one side surface of the cartridge tray 2 so as to project outwardly. There is a loader drive groove 5-1 in the side surface of the loader drive plate 5, and the loader drive groove 5-1 has a slant portion 5-1 a which is higher towards the front and lower towards the rear with respect to the X-direction, and a horizontal portion 5-1 b connected to the end of the slant portion 5-1 a in the Y-direction. There are loader guide grooves 4-1 and 4-2 in the side surface of the loader guide plate 4. The pin 2-1 extends so as to penetrate through the loader drive groove 5-1 and the loader guide groove 4-1. A roller 2-2, which is movable in and along the loader drive groove 5-1, and a roller 2-3 which is movable in and along the loader guide groove 4-1, are rotatably secured to the pin 2-1. These rollers 2-2 and 2-3 have flanges serving as a spacer between the side surface of the cartridge

tray 2 and the side surface of the loader drive plate 5, and a spacer between the side surface of the loader drive plate 5 and the side surface of the loader guide plate 4. There is a pin 5-2 on the side surface of the loader drive plate 5 which projects outwardly, and a roller 5-3 which is movable in and along the loader guide groove 4-2 and which is rotatably secured to the pin 5-2.

**[0037]** In Fig. 7. the relationship between the loader guide plate 4, the loader drive plate 5 and the cartridge tray 2 is shown with respect to one side surface side. The same relationship is satisfied with respect to the other side surface side. However, as shown in Figs. 1 and 2, in addition to the pin 2-1, there is a similar pin 2-1' on the other side surface of the cartridge tray 2, which projects outwardly. In connection with the pin 2-1', there is a loader drive groove 5-1', as well as the loader drive groove 5-1, in the other side surface of the loader drive plate 5. The pin 2-1' extends so as to penetrate through the loader drive groove 5-1' and the loader guide groove 4-2, and a roller 2-2' which is movable in and along the loader drive groove 5-1', and a roller 2-3' which is movable in and along the loader guide groove 4-2, are rotatably secured to the pin 2-1' as in the case of the pin 2-1. Further, as shown in Figs. 1 and 2, in addition to the pin 5-2, there is a pin 5-2' on the other side surface of the loader drive plate 5 arranged to project outwardly, and a roller 5-3' which is movable in and along the loader guide groove 4-2 is rotatably secured to the pin 5-2'.

**[0038]** As shown in Fig. 2, each of the loader guide grooves 4-1, 4-2 has a horizontal portion extending in the X-direction and a vertical portion which extends substantially vertically downwardly from the end of the horizontal portion in the X-direction. The shapes of the loader guide grooves 4-1 and 4-2, and the positions of the pins 2-1 and 2-1', are such that the rollers 2-3, 2-3' arrive at the vertical portions of the loader guide grooves 4-1, 4-2 at the same time.

**[0039]** With the above construction, the loader drive plate 5 and the cartridge tray 2 can be mounted on the loader guide plate 4 so as to be movable in the X and Y directions and in the direction perpendicular to the X and Y directions.

**[0040]** A loader drive roller 5-4 is secured to the upper surface of the loader drive plate 5, and it extends to the upper portion of the loader guide plate 4 so as to pass through a groove 4-3 on the upper surface of the loader guide plate 4 along the X and Y directions as shown in Fig. 1.

**[0041]** As shown in Fig. 7, an engaging member 2-4 is formed on the cartridge tray 2 so as to be engageable with the engaging projection 1-4 provided to the door of the cartridge 1, and when the cartridge 1 is inserted in a predetermined position of the tray 2, the engaging projection 1-4 and the engaging member 2-4 are engaged with each other to open the cartridge door 1-3.

**[0042]** An external loader drive gear 6 is disposed at the upper side of the loader guide plate 4 so as to be rotatable around the up-and-down direction relative to

the deck base 3. The loader drive gear 6 is driven through a gear train 23 by a loader motor 22 serving as the driving means which is rotatable in either the forward or the reverse direction, as shown in Fig. 6.

**[0043]** There is a loader drive cam groove 6-1 in the lower surface of the loader drive gear 6, and the loader drive roller 5-4 is fitted to the cam groove 6-1. The cam groove 6-1 has a radial-direction portion 6-1a extending in the radial direction of the loader drive gear 6, and a circumferential-direction portion 6-1b which is connected to the outer end of the radial-direction portion 6-1a and extends in the circumferential direction. As shown in Fig. 18, the groove width of the radial-direction portion 6-1a of the cam groove is set to be slightly larger than the outer diameter of the loader drive roller 5-4, except for the connection end portion with the circumferential-direction portion 6-1b of the cam groove. A spring 6-2 is secured at the X-direction side of the radial-direction portion 6-1a of the cam groove on the lower surface of the loader drive gear 6, and the spring 6-2 biases the loader drive roller 5-4 located in the radial-direction portion 6-1a in the Y direction. Accordingly, even when the loader drive gear 6 is still, the loader drive plate 5 can move slightly in the X direction against the spring 6-2 as a result of the external force while the loader drive roller 5-4 is located in the radial-direction portion 6-1a of the cam groove. That is, in Figs. 1A and 1B, the spring 6-2 behaves as if it forms an "X-direction movable side wall biasing in the Y direction" of the radial-direction portion 6-1a of the cam groove. In the figures, other than Figs. 1 A and 1 B, the spring 6-2 is omitted from the illustration.

**[0044]** In Fig. 1A and other figures, the cam groove 6-1 is illustrated by a solid line as if it extends so as to penetrate to the upper surface of the loader drive gear 6. However, this illustration is made for emphasis. Actually, the cam groove 6-1 is only at the lower surface side of the loader drive gear 6, as shown in Fig. 6.

**[0045]** Further, the face gear 21 and the driving motor 24 for driving the face gear 21 shown in Fig. 3 are arranged below the loader guide plate 4, the loader drive plate 5 and the cartridge tray 2, so as to be fixed in position to the deck base 3.

**[0046]** A threadder drive arm 7 is arranged in the threading mechanism portion so as to be rotatable around the up-and-down direction with the pin 25 at the centre of the rotation with respect to the deck base 3, as shown in Figs. 1A and 2. A threadder drive roller 7-1 is secured on the upper surface of the threadder drive arm 7 so as to be away from the pin 25 and rotatable around the up-and-down direction. A threading arm 8 is secured to the tip portion of the threadder drive arm 7 so as to be rotatable around the up-and-down direction. The threadder drive arm 7 and the threading arm 8 constitute the threadder drive (arm) member.

**[0047]** As shown in Fig. 8, a pin 26 extending in the up-and-down direction is secured to the tip portion of the threading arm 8, and one end portion of the leader block 9 is secured to the pin 26 at a lower position than the

threading arm 8. The pin 26 has a flange portion at its lower end, and any downward movement of the leader block 9 is prevented by the flange portion. That is, as shown in Fig. 9, the leader block 9 is provided with a through hole 9-1 which extends in the up-and-down direction and has a composite shape containing a small circle and a large circle in section. The pin 26 is designed so that the outer diameter of the portion thereof which penetrates through the through hole 9-1 is smaller than the diameter of the small circle, and the outer diameter of the flange portion at the lower end thereof is larger than the diameter of the large circle. Accordingly, the leader block 9 is loosely fitted to the pin 26 and held by the pin 26 so as to be rotatable around the pin 26.

**[0048]** The leader block 9 has a leader pin capture groove 9-2 on the side surface of the other end portion, the leader pin capture groove 9-2 extending in the up-and-down direction and being fitted to the leader pin 1-7 affixed to the tip of the magnetic tape 1-6 to capture the leader pin 1-7. A downwardly projecting pin 9-3 and an upwardly projecting pin 9-4 are secured to the leader block 9.

**[0049]** Referring to Figs. 1A and 2, a threading cam 10 is arranged horizontally at a position higher than the threading arm 8. There is a threading cam groove 10-1 on the lower surface of the threading cam 10. To the threading cam groove 10-1 is fitted a threading guide roller 27 which is secured to the upper end of the pin 26 which in turn is secured to the tip of the threading arm 8, as shown in Fig. 8.

**[0050]** As shown in Figs. 1A and 2, an external threader drive gear 11 is arranged at the upper side of the threading cam 10 so as to be rotatable around the up-and-down direction relative to the deck base 3. The threader drive gear 11 is engaged with the loader drive gear 6, and rotated in the reverse direction to the loader drive gear 6 when the loader drive gear rotates.

**[0051]** A threader drive cam groove 11-1 is formed on the lower surface of the threader drive gear 11, and the threader drive roller 7-1 is fitted to the cam groove 11-1. The cam groove 11-1 has a radial-direction portion 11-1a extending in the radial direction of the threader drive gear 11 and a circumferential-direction portion 11-1 b which is connected to the inner end of the radial-direction portion 11-1a and extends in the circumferential direction.

**[0052]** In the Figures, the cam groove 11-1 and the threading cam groove 10-1 are illustrated with solid lines (or illustrated with one solid line) as if they extended to the upper surface of the threader drive gear 11 and the upper surface of the threading cam 10 respectively. However, this illustration is for the purpose of emphasis, and the cam grooves 11-1 and 10-1 are actually formed only at the lower surface sides of the threader drive gear 11 and the threading cam 10 respectively, as in the case of the loader drive cam groove 6-1.

**[0053]** Further, a driving motor 30 which is fixed to the deck base 3 and a machine reel 31 which is rotationally driven around the up-and-down direction are arranged

at the lower side of the threader drive arm 7 and the threading cam 10.

**[0054]** As shown in Fig. 1A, at the outside of the machine reel 31 in the radial direction there are plural tape guide rollers 32 and a magnetic head 34 arranged between the machine reel 31 and the threading cam groove 10-1, and located along the tape running route inside the tape running route (at the machine reel side).

**[0055]** As shown in Fig. 10, the leader block 9 is fitted to a retractor block 12. The retractor block 12 has a pin 12-1 extending in the up-and-down direction at the lower portion thereof, and disposed so as to be rotatable around the up-and-down direction with the pin 12-1 at the centre of the rotation with respect to the deck base 3. A downwardly projecting retractor pin 12-2 is formed at the lower portion of the retractor block 12 so as to be away from the rotational centre pin 12-1. The retractor pin 12-2 is located adjacent to the moving route of the cartridge tray 2, and it can be engaged with a retractor cam 5-5 (see Fig. 10) provided on the side surface of the loader drive plate 5 when the cartridge tray 2 is moved in the X direction.

**[0056]** One end of a retractor spring 13 is connected to the side portion of the retractor block 12, and the other end of the spring 13 is fixed to the deck base 3. A linear groove 12-3 is formed on the upper surface of the retractor block 12, and the pin 9-3 of the leader block 9 is fitted to the groove 12-3 so as to be displaced from the pin 12-1.

**[0057]** The leader block 9 is fitted to a horizontally arranged guide cam 33, as shown in Fig. 10. The guide cam 33 is fixed to the deck base 3, and has a cam groove 33-1. The pin 9-4 of the leader block 9 is fitted in the cam groove 33-1. The cam groove 33-1 has an arcuate portion 33-1a having an arcuate shape of substantially 90 degrees and a linear portion 33-1 b having a substantially linear shape which has one end connected to one end of the arcuate portion 33-1 a and the other end which is opened to the outside and which extends substantially in the radial direction of the arcuate shape. With this construction, the rotation of the leader block 9 around the pin 26, when the pin 26 is located at a predetermined position as shown in Fig. 10, is restricted to a predetermined range (that is, the movement of the pin 9-4 of the leader block 9 along the arcuate portion 33-1 a of the cam groove enables the leader block 9 to rotate in an angular range of substantially 90 degrees).

**[0058]** Fig. 11 is a schematic plan view showing the machine reel 31. A notch 31-2 having a shape corresponding to the leader block 9 is formed in the core portion of the machine reel 31, and an upper side flange portion 31-3 of the machine reel 31 has a groove 31-4 which is connected to the notch 31-2 of the core portion at the inner periphery of the upper side flange portion 31-3 and which extends to the outer periphery. Accordingly, the leader block 9 is fitted into the notch 31-2 of the core portion of the machine reel while the leader pin 1-7 is captured in the leader pin capture groove 9-2, as shown in Fig. 11.

**[0059]** Next, the operation of the mechanism as described above will be described.

**[0060]** First, the cartridge 1 is inserted into the cartridge tray 2 in the X-direction either manually, or by using a handling apparatus, as shown in Figs. 1A, 2, and 7. Figs. 1A and 2 show the situation in which the cartridge insertion has been completed. In this situation, the cartridge 1 is allowed to be discharged (ejected) from the loading mechanism portion. The position of the cartridge tray 2 in this condition is called the "eject position".

**[0061]** When the cartridge 1 is inserted into the cartridge tray 2, the fitting of the cartridge 1 to the cartridge tray 2 is carried out at the time when the cartridge tray 2 arrives at the eject position. That is, as shown in the partial plan views of Figs. 12A and 12B, a lock block 36 is secured to the side surface of the cartridge tray 2 so as to be rotatable around the pin 35 extending in the up-and-down direction. The lock block 36 has a hook portion 36-1 formed at the tip thereof, and also a cam face portion 36-2 formed on the upper surface thereof. There is an opening 2-5, through which the hook portion 36-1 passes, in the cartridge tray 2 at the position corresponding to the hook portion 36-1 of the lock block. The lock block 36 is biased to the side surface of the cartridge 1 in the direction of an arrow k by biasing means (not shown). In the side surface of the cartridge 1 there is a lock hole 1-9 arranged so as to be approachable at the position corresponding to the tip of the lock block 36.

**[0062]** In the process in which the cartridge 1 is inserted into the cartridge tray 2, the spring 6-2 biases the loader drive roller 5-4 in the Y direction, as shown in Fig. 1B, and thus the lock block 36 is located so that the cam face portion 36-2 is fitted to a pin 37 which is fixed to the deck base 3, as shown in Fig. 12A. In this state, the lock block 36 is rotated in the opposite direction by a predetermined angle against the biasing force in the k direction, and the hook portion 36-1 is retracted from the opening 2-5, so that the cartridge 1 can be inserted into the cartridge tray 2 in the X direction without suffering interference by the hook portion 36-1.

**[0063]** After the cartridge 1 has been inserted deeply into the cartridge tray 2, the cartridge 1 can be pushed slightly further in the X direction. This is because the loader drive roller 5-4, and thus the loader drive plate 5 can be slightly moved in the X direction against the biasing force of the spring 6-2 relative to the loader drive gear 6 at the eject position as described with reference to Figs. 1A and 1B. Upon the movement of the cartridge 1 and the cartridge tray 2 in the X direction, the lock block 36 arrives at a position in which the fitting of the cam face portion 36-2 to the pin 37 is released. In this situation, as shown in Fig. 12B, the lock block 36 is rotated by the biasing force in the k direction, so that the hook portion 36-1 at the tip of the lock block 36 is passed through the opening 2-5, inserted into the lock hole 1-9 of the cartridge 1 and locked there, as shown in Fig. 12B. In this situation, the loader drive roller 5-4 is slightly separated from the wall in the Y-direction side in the radial-direction

portion 6-1A of the cam groove.

**[0064]** Accordingly, by moving the cartridge tray 2 in the X direction, the cartridge 1 can also subsequently be moved integrally in the X direction. The loading/threading operation using the driving force of the loader motor 22 can be started by detecting the insertion/lock of the cartridge 1 into/to the cartridge tray 2 at the eject position.

**[0065]** When the cartridge 1 is discharged from the cartridge tray 2 in the Y direction at the eject position, the 10 condition shown in Fig. 12B is changed to that shown in Fig. 12A. That is, the lock block cam face portion 36-2 is fitted to the pin 37 which is positionally fixed to the deck base 3, whereby the lock block 36 is rotated in the opposite direction against the biasing force in the k direction, the fitting of the hook portion 36-1 and the lock hole 1-9 is released and the cartridge 1 is allowed to be discharged from the cartridge tray 2 in the Y direction. At this time, the loader drive roller 5-4 is pushed in the Y direction by the fixed wall at the X-direction side of the cam groove 20 radial-direction portion 6-1 a at the connection end portion with the cam groove circumferential-direction portion 6-1 b, thereby carrying-out the movement of the lock block 36 in the Y-direction from the pin 37.

**[0066]** At the eject position, the engaging projection 25 1-4 of the door of the cartridge 1 is engaged with the engaging member 2-4 of the cartridge tray 2 to keep the door of the cartridge 1 open, as described with reference to Fig. 7.

**[0067]** At the eject position, the loader drive roller 5-4 30 is engaged with the radial-direction portion 6-1 a of the loader drive cam groove 6-1, and the threader drive roller 7-1 is engaged with the circumferential-direction portion 11-1 b of the threader drive cam groove 11-1, as shown in Figs. 1A and 1B.

**[0068]** Further, at the eject position, the pin 2-1, 2-1' 35 is located at the tip in the X-direction in the slant portion of the loader drive groove 5-1, 5-1' so as to be away from the vertical portion of the loader guide groove 4-1, 4-2 at a predetermined distance in the Y direction.

**[0069]** A lock mechanism for the cartridge tray 2 is set 40 so that only when the cartridge 1 is inserted in the cartridge tray 2 in an accurate direction, is the subsequent movement of the cartridge tray 1 in the X-direction allowed.

**[0070]** That is, as shown in Fig. 13A, a lock lever 40 is 45 secured to the cartridge tray 2 so as to be rotatable around a pin 39 extending in the up-and-down direction. The lock lever 40 is biased in the direction of an arrow m by appropriate means (not shown). An engaging portion 40-1 is formed on the lock lever 40, and it is engaged with a pin 41 which is fixed in position to the deck base 3, thereby preventing the movement of the cartridge tray 2 in the X direction. As shown in Fig. 13A this situation continues until the tip of the lock lever 40 abuts against 50 the slant side wall surface (side wall surface located obliquely to the X direction) 1-10 formed by cutting out a corner portion of the upper surface of the cartridge 1.

**[0071]** When the cartridge 1 is further moved in the X-

direction from the condition shown in Fig. 13A relatively to the cartridge tray 2, the slant side wall surface 1-10 of the cartridge abuts against the tip of the lock lever 40 to rotate the lock lever against the m-direction biasing force in the opposite direction, whereby the engagement of the lock lever engaging portion 40-1 and the pin 41 is released, and the movement of the cartridge tray 2 in the X-direction relative to the deck base 3 is allowed. Fig. 13B shows an eject-allowed state.

**[0072]** The pin 41 shown in Figs. 13A and 13B is arranged at such a height that the movement of the cartridge tray 2 in the X and Y directions is not disturbed.

**[0073]** When the cartridge 1 is inserted into the cartridge tray 2 in an incorrect direction, the lock release, based on the engagement between the slant side wall surface 1-10 of the cartridge and the tip portion of the lock lever 40, as described above, is not carried out.

**[0074]** Subsequently, the loader drive gear 6 is rotated in the direction of the arrow A by a predetermined angle to change the condition of the mechanism from the ejection state shown in Figs. 1A and 2 to that shown in Figs. 14 and 15. In such a process, the cartridge 1 is moved in the X-direction, together with the cartridge tray 2 by a predetermined distance. The loader drive roller 5-4 is still engaged with the radial-direction portion 6-1 a of the loader drive cam groove 6-1, as shown in Fig. 14, while the threader drive roller 11 is engaged with the circumferential-direction portion 11-1b of the threader drive cam groove 11-1.

**[0075]** Further, in this state, the pins 2-1, 2-1' are located at the connection position between the slant portion and the horizontal portion of the loader drive grooves 5-1, 5-1', and located at the lowest position of the vertical portion of the loader guide grooves 4-1, 4-2.

**[0076]** That is, the cartridge tray 2 as well as the cartridge 1 are moved downwardly and located at a position lower than the position corresponding to the ejection state. In this situation, the cartridge reel in the cartridge 1 and the machine reel 31 are located at the same height. As described with reference to Fig. 3, the downward movement of the cartridge 1 allows engagement to take place between the face gear 20 of the cartridge reel 1-2 and the face gear 21 at the driving motor side and releases the lock of the cartridge reel 1-2.

**[0077]** In the situation shown in Figs. 14 and 15, the standby of the threading operation of the magnetic tape in the cartridge 1 by the threading mechanism portion has been completed, and the rotation of the cartridge reel by the driving motor 24 is allowed. The position of the cartridge tray 2 in this situation, is referred to as the "mount position".

**[0078]** In the retracted unload position of the threading operation, the leader block 9 is kept engaged with the retractor block 12 until the mount state of the loading mechanism portion is established. The position of the leader block 9 in the situation in which the leader block 9 is engaged with the retractor block 12 is referred to as the "unload position".

**[0079]** The unload position, particularly in the situation in which the leader block 9 is arranged substantially along the X-Y direction by the biasing force of the retractor spring 13, and is retracted from the cartridge movement route so that it does not obstruct the movement of the cartridge 1 as shown in Figs. 1A, 10, and 14 is referred to as the "retracted position".

**[0080]** Subsequently, in the leader pin captured position, the loader drive gear 6 is rotated in the direction of the arrow A by a predetermined angle to change the state of the mechanism from the condition shown in Figs. 14 and 15 to the condition shown in Figs. 16 and 17. In the resulting situations, the cartridge 1 and the cartridge tray 2 are kept in the retracted positions, however, the loader drive plate 5 is slightly advanced in the X-direction from the retracted position. That is, the pins 2-1, 2-1' are located at the Y-direction end of the horizontal portion of the loader drive grooves 5-1, 5-1'.

**[0081]** In this situation, as shown in Fig. 16, the loader drive roller 5-4 is engaged with the connection portion between the radial-direction portion 6-1 a and the circumferential-direction portion 6-1 b of the loader drive cam groove 6-1, and the threader drive roller 11 is engaged with the circumferential-direction portion 11-1b of the threader drive cam groove 11-1.

**[0082]** In this situation, as shown in Fig. 18, the tip portion of the leader block 9 invades from the opening of the cartridge 1 into the cartridge, and the leader pin capture groove 9-2 captures the leader pin 1-7 located at a predetermined position in the cartridge 1. The position of the leader block 9 corresponding to this state is referred to as the "leader pin captured position".

**[0083]** That is, in the shift progress from the retracted position to the leader pin captured position, the retractor cam 5-5 (see Figs. 10, 18) fixed to the loader drive plate 5 is engaged with the retractor pin 12-2, and the retractor block 12 is rotated around the pin 12-1 in the clockwise direction of Fig. 18 by about 90 degrees against the tensile force of the spring 13, whereby the leader block 9 which is engaged with the groove 12-3 of the retractor block 12 through the pin 9-3 is rotated around the pin 26 by about 90 degrees. When the leader block 9 is rotated, the pin 9-4 is moved from the deepest position in the cam groove arcuate portion 33-1 a of the guide cam 33 shown in Fig. 19A to the connection portion between the arcuate portion 33-1A and the linear portion 33-1 b of the cam groove shown in Fig. 19B.

**[0084]** As shown in Fig. 18, the rotation of the leader block 9 by substantially 90 degrees makes the tip portion of the leader block invade from the opening of the cartridge 1 into the cartridge, whereby the leader pin 1-7 located at a predetermined position in the cartridge 1 is captured by the leader pin capture groove 9-2.

**[0085]** Subsequently, in the load position, the loader drive gear 6 is rotated in the direction of the arrow A by a predetermined angle to change the state of the mechanism from the situation shown in Figs. 16 and 17 to the situation shown in Fig. 20 and then to the situation shown

in Fig. 21. In the resulting situations, the loader drive roller 5-4 is engaged with the circumferential-direction portion 6-1 b of the loader drive cam groove, and thus the X-direction positions of the loader drive plate 5, the cartridge 1 and the cartridge tray 2 are identical to those of the leader pin captured position.

**[0086]** In the situation shown in Fig. 21, the leader block 9 is fitted to the core portion of the machine reel 31. The position of the leader block in the situation in which the leader block 9 is fitted to the machine reel 31 is referred to as the "load position".

**[0087]** In the shift progress from the leader pin captured position to the load position, the threader drive roller 7-1 is first located in the circumferential-direction portion 11-1 b of the threader drive cam groove, and thus the threader drive arm 7 and the threading arm 8 stand still (the threader drive roller 7-1 runs idle). During this idle period, the standby for the subsequent threading operation, for example, the application of the tape tension and other operations can be carried out.

**[0088]** Since the threader drive roller 7-1 will soon be located in the radial-direction portion 11-1 a of the threader drive cam groove, the threader drive arm 7 is rotated around the pin 25 thereby to produce horizontal tensile force to the leader block 9 through the threading arm 8 and the pin 26 secured to the tip of the threading arm 8.

**[0089]** Since the threading guide roller 27 secured to the pin 26 is engaged with the threading cam groove 10-1, as shown in Fig. 8, the leader block 9 starts to move along the threading cam groove 10-1 to the machine reel 31.

**[0090]** When the movement of the leader block 9 starts, the groove 12-3 of the retractor block 12 is set to be substantially perpendicular to the X-direction as shown in Fig. 18, and the linear portion 33-1 b of the cam groove 33-1 of the guide cam 33 is substantially perpendicular to the X-direction as shown in Fig. 19B. Therefore, the pin 9-3 at the lower side of the leader block 9 and the pin 9-4 at the upper side thereof can be moved in a direction substantially perpendicular to the X-direction, so as to move along the groove 12-3 and the cam groove portion 33-1.

**[0091]** Accordingly, the engagement of the leader block 9 with the retractor block 12 and the guide cam 33 is released, and the leader block 9 is moved along the threading cam groove 10-1 while held by the pin 26. During the movement of the leader block 9, the magnetic tape 1-6 is drawn out of the cartridge 1. Fig. 20 shows the situation in which the leader block 9 is being fed at some midpoint.

**[0092]** Under the load situation of Fig. 21, the leader block 9 invades into the notch 31-2 of the core portion of the machine reel 31 (see Fig. 11). As shown in Fig. 21, the magnetic tape 1-6 thus drawn out is tensed along the tape guide roller 32 by the tensile force which is applied to the magnetic tape when the magnetic tape is drawn out of the cartridge 1, thereby allowing the magnetic head 34 to make access to the magnetic tape.

**[0093]** As shown in Fig. 8, the upper surface of the lower-end flange portion of the pin 26 is tapered, and in connection with this tapered structure, a taper surface is formed at the lower end portion of the leader block 9 through hole 9-1. Accordingly, when the leader block 9 is moved, the taper surface at the lower end portion of the pin 26 and the tapered upper surface of the flange portion at the lower end of the pin 26 are engaged with each other to keep the leader block 9 at a predetermined height, whereby the tension of the magnetic tape can be well maintained.

**[0094]** The rotational position of the machine reel 31 is set so that the groove 31-4 of the flange portion of the machine reel 31 and the threading cam groove 10-1 are lined up before the leader block 9 arrives at the machine reel 31. The setting of the rotational position may be achieved by using a combination of electrical signals obtained from photo-electric means, such as a photo-interrupter for detecting light intercepting walls which are formed along the circumferential direction on the upper or lower surface of the threader drive gear 11, so as to have a starting end and a terminating end. With this operation, the pin 26 supporting the leader block 9 can pass through the groove 31-4 of the flange portion of the machine reel 31 and advance into the notch 31-2 of the core portion.

**[0095]** After the tape threading operation as described above has been completed, the machine reel 31 is driven by the motor 30. Figs. 22A and 22B show the variation of the relationship between the machine reel core portion 31-1 and the leader block 9 at that time. That is, before the machine reel is driven, the pin 26 is located at the rotational centre of the machine reel and brought into contact with the inner surface of the through hole 9-1 of the leader block 9, as shown in Fig. 22A. The end portion (the end portion at the leader pin capture groove side) of the leader block 9 at the opposite side to the through hole 9-1 is located so as slightly to project from the outer periphery of the reel core portion 31-1.

**[0096]** Thereafter, when the machine reel 31 is rotated in the direction of an arrow B by the motor 30, due to the tensile force applied from the machine reel core portion 31-1 to the tape, the end portion of the leader pin capture groove side of the leader block 9 is pressed inwardly and slightly moved towards the core centre, whereby the through hole 9-1 of the leader block and the pin 26 are kept out of contact with each other. Therefore, the leader block 9 can be rotated integrally with the reel core by the motor 30 while the pin 26 is disposed in the through hole 9-1 having a larger inner diameter than the outer diameter of the pin 26.

**[0097]** Thereafter, the cartridge reel and the machine reel are rotated in desired directions in a desired manner to carry out an information recording/reproducing operation on the magnetic tape 1-6 through the magnetic head 34.

**[0098]** When the magnetic tape 1-6 is returned to the cartridge reel 1-2 after the information recording/repro-

ducing operation, the cartridge reel 1-2 and the machine reel 31 are first set to the position as shown in Fig. 21 by the reel motors 24, 30, and then the loader motor 22 is rotated in the opposite direction to that which exists when the tape threading is carried out. At this time, the reel motor 24 is also rotated at a proper speed in a predetermined direction so that the proper tensile force is applied to the magnetic tape 1-6, whereby the leader block 9 is moved from the load position to the unload position (leader pin captured position), and the leader pin 1-7 arrives at the leader pin accommodation portion 1-8 of the cartridge 1. Subsequently, the rotation of the reel motor 24 is stopped, and the leader block 9 is moved from the leader pin captured position to the retracted position by continuing the rotation of the loader motor 22. Subsequently, the cartridge tray 2 is moved from the mount position to the eject position, whereby the reverse operation to that employed when the magnetic tape is drawn out is carried out in each part.

**[0099]** In the loading/threading mechanism of the above-described arrangement, a sequential operation of mounting the magnetic tape cartridge on to the magnetic tape apparatus, drawing the magnetic tape leading end out of the magnetic tape cartridge and connecting the magnetic tape leading end to the machine reel can be carried out by a single driving force generating source. Accordingly, the loading/threading operation can be sufficiently detected by detecting only the rotational angular position of the loader drive gear or the threader drive gear, and thus the need for a number of sensors for detecting the state of the mechanism and for controlling the apparatus can be reduced. Further, it is possible by means of this arrangement to reduce the number of driving force generating sources. Therefore, the control circuit of the apparatus can be simplified, and the apparatus can be made smaller in size and its cost reduced.

**[0100]** Fig. 23 is a plan view showing another arrangement of a loading/threading mechanism of a cartridge magnetic tape apparatus. In Fig. 23, the elements or parts having the same functions as those described with reference to Fig. 1A to Fig. 22B are indicated by the same reference numerals.

**[0101]** In this arrangement, the loader drive cam groove and the threader drive cam groove are on the same driving gear 15. The threader drive cam groove has a radial-direction portion 11-1 a and a circumferential-direction portion 11-1 b similar to those of the first-described arrangement, however, the loader drive cam groove has a circumferential-direction portion 6-1 b similar to that of the first-described arrangement and a radial/circumferential direction portion 6-1c different from that of the first-described arrangement. The radial/circumferential direction portion 6-1c is a non-circumferential direction groove portion having a radial direction component and a circumferential-direction component, and the inner end thereof is connected to the circumferential-direction portion 6-1 b while the outer end thereof is located in the neighbourhood of the outer periphery of the driving

gear 15.

**[0102]** In Fig. 23, there are shown the eject position of the cartridge tray and a cartridge 1' at the mount position.

**[0103]** The driving gear 15 is driven by a driving motor (not shown) similar to the loader drive motor of the first-described arrangement. Following the rotation of the driving gear 15, the loader drive roller 5-4 is moved in the radial/circumferential direction portion 6-1 c to the circumferential direction portion 6-1 b on the basis of the engagement between the loader drive roller 5-4 and the radial/circumferential direction portion 6-1c of the loader drive cam groove, and also moved in the X-direction relative to the deck base, thereby carrying out the movement of the cartridge tray from the eject position to the mount position and the subsequent movement of the leader block from the retracted position to the leader pin captured position as carried out in the first-described arrangement. The subsequent operation is the same as in the first-described arrangement.

**[0104]** In this second arrangement, the same effect can be achieved as in the first-described arrangement, and also the apparatus can be further be made smaller, because of the use of the single driving gear 15 as a driving rotator.

**[0105]** In the above arrangement, the gear is used as the driving rotator or the driving rotating member. However, a pulley, a roller or other rotator or rotating members may be used. Further, the linkage between a first rotator and a second rotator may be provided by the suspension of a chain or belt around these rotators or by pressing the rotators upon each other.

**[0106]** It will be understood that although particular arrangements have been described, by way of example in illustration of the invention, variations and modifications thereof, as well as other arrangements can be conceived within the scope of the appended claims.

## Claims

1. A loading/threading mechanism for a single reel cartridge magnetic tape apparatus, including a loading mechanism (4-6) for moving a cartridge tray (2) for accommodating a cartridge (1) between an eject position in which the cartridge is mountable/demountable into/from the cartridge tray and a mount position in which a reel (1-2) of the cartridge is engaged with a cartridge reel rotating means (21) of the magnetic tape apparatus, and a threading mechanism (7-13, 26, 27, 33) for moving the tip portion of a magnetic tape (1-6) accommodated in the cartridge between a position in the cartridge tray located at the mount position and a position in a reel (31) of the magnetic tape apparatus, the loading mechanism and the threading mechanism being driven by means of a single driving force generating source (22, 23); and characterized in that:

the threading mechanism includes a leader block (9) which is detachably engageable with a leader pin (1-7) affixed to the tip portion of the magnetic tape, the threading mechanism being adapted such that

the leader block is moved through a predetermined route between an unload position in the neighbourhood of the cartridge tray located at the mount position and a load position in a notch (31-2) of the core portion of the reel of the magnetic tape apparatus; and at the unload position, the leader block is located either at a retracted position at which the leader block is disengaged from the leader pin and retracted from the cartridge accommodated in the cartridge tray located at the mount position, or at a leader pin captured position at which the leader block is engaged with the leader pin affixed to the tip portion of the magnetic tape in the cartridge accommodated in the cartridge tray located at the mount position.

2. A loading/threading mechanism as claimed in Claim 1, wherein the loading mechanism has a loader drive rotator having a loader drive cam groove (6-1), and a loader drive member which is reciprocally movable in one direction to translate the cartridge tray (2) between the eject position and the mount position, the loader drive member having a loader drive engaging member which is engaged with the loader drive cam groove, the threading mechanism having a threader drive rotator with a threader drive cam groove (11-1) and a threader drive member for driving the leader block (9) to move between the unload position and the load position, the threader drive member having a threader drive engaging member for engaging with the threader drive cam groove, and the loader drive rotator and the threader drive rotator being connected to each other, so that a rotational driving force is transmitted therebetween, and a driving force is transmitted from the driving force generating source to one of the loader drive rotator and the threader drive rotator.
3. A loading/threading mechanism as claimed in Claim 1, wherein the loading mechanism has a loader drive cam groove (6-1) formed in a driving rotator, and a loader drive member which is reciprocally movable in one direction to translate the cartridge tray (2) between the eject position and the mount position, the loader drive member having a loader drive engaging member for engagement with the loader drive cam groove, the threading mechanism having a threader drive cam groove (11-1) in the driving rotator and a threader drive member for driving the leader block (9) to move between the unload position and the load position, and the threader drive member having a threader drive engaging member for engagement

with the threader drive cam groove, whereby a driving force may be transmitted from the driving force generating source to the driving rotator.

5. 4. A loading/threading mechanism as claimed in either of Claim 2 or Claim 3, wherein the threading mechanism has a retractor (12) for moving the loader drive member in the condition in which the cartridge tray (2) is in the mount position, thereby shifting the leader block (9) from the retracted position to the leader pin captured position, and the retractor having a pin (12-2) which is engageable with a retractor cam (5-5) secured to the loader drive member, and a groove (12-3) which is engageable with a pin (9-3) secured to the leader block.
5. A loading/threading mechanism as claimed in Claim 1, wherein the loading mechanism and the threading mechanism include a loader drive cam groove (6-1) and a threader drive cam groove (11-1) on a rotating member respectively, the loader drive cam groove having a first groove portion (6-1 b) extending in the circumferential direction so as to be kept away from the rotational centre of the rotating member at a fixed distance, and a second groove portion (6-1a) which is linked to one end portion of the first groove portion and extends so that the distance from the rotational centre of the rotating member is varied, the loader drive cam groove being engaged with a loader drive engaging member affixed to a loader drive member, the loader drive member being engaged with the cartridge tray (2), whereby in the condition in which the second groove portion is engaged with the loader drive engaging member, during the rotation of the rotating member the cartridge tray is moved between the eject position and the mount position, the threader drive cam groove having a third groove portion (11-1 b) extending in the circumferential direction so as to be kept away from the rotational centre of the rotating member at a fixed distance and a fourth groove portion (11-1a) which is linked to one end portion of the third groove portion and extends so that the distance from the rotational centre of the rotation member is varied, the threader drive cam groove being engaged with a threader drive engaging member affixed to a threader drive member, the threader drive member being connected to the leader block (9) whereby in the condition in which the fourth groove portion is engaged with the threader drive engaging member, during the rotation of the rotating member the leader block is moved through the predetermined route, the first to fourth groove portions being set so that when the loader drive engaging member is engaged with the second groove portion, the threader drive engaging member is engaged with the third groove portion, and, when the threader drive engaging member is engaged with the fourth groove portion, the loader drive engaging

member is engaged with the first groove portion, and the single driving force generating source transferring the rotational force to the rotating member.

6. A loading/threading mechanism as claimed in Claim 5, wherein the rotating member has a first rotator and a second rotator which are connected to each other so as to be rotatable in synchronism with each other, the loader drive cam groove (6-1) being in the first rotator, and the threader drive cam groove (11-1) being in the second rotator. 5

7. A loading/threading mechanism as claimed in Claim 6, wherein the first rotator is a loader drive gear (6), and the second rotator is a threader drive gear (11), the loader drive gear and the threader drive gear being engaged with each other. 10

8. A loading/threading mechanism as claimed in Claim 5, wherein the loader drive member moves the cartridge tray (2) while a guide engaging member affixed to the cartridge tray is guided in engagement with a loader guide groove (4-1,4-2) formed in a loader guide member. 15

9. A loading/threading mechanism as claimed in Claim 8, wherein the loader guide groove has a bend in it. 20

10. A loading/threading mechanism as claimed in any one or more of Claims 5 to 9, wherein the leader block (9) is located at one of the retracted position and the leader pin captured position in the unload position. 25

11. A loading/threading mechanism as claimed in Claim 10, including a retractor block (12) which is rotatable to rotate the leader block (9) located at the unload position, the retractor block being biased in one rotational direction, the loader drive member having a retractor cam (5-5) which is engaged with the retractor block to rotate the retractor block in the other rotational direction, and the loader drive member being moved while the cartridge tray (2) is located at the mount position, whereby the leader block is rotated against the biasing force to be shifted from the retracted position to the leader pin captured position. 30

12. A loading/threading mechanism as claimed in Claim 11, including a guide cam (33) for restricting the range of the angle of rotation of the retractor block at the unload position. 35

13. A loading/threading mechanism as claimed in Claim 5, wherein the cartridge tray (2) has an engaging member (2-4) for engagement with a door (1-3) of the cartridge inserted at the eject position to open the door. 40

14. A loading/threading mechanism as claimed in Claim 5, wherein the cartridge tray (2) has a lock member which is engaged with a lock hole (1-9) of the cartridge inserted to the eject position to fix the cartridge to the cartridge tray. 45

15. A loading/threading mechanism as claimed in Claim 5, including a lock mechanism for fixing the cartridge tray (2) at the eject position, the lock mechanism abutting against the slant surface of the cartridge to release the fixing. 50

16. A loading/threading mechanism as claimed in Claim 5, wherein the leader block (9) is rotatably held with a clearance by a support pin secured to the threader drive member, and including a threading cam groove for guiding the support pin when the leader block is moved between the unload position and the load position. 55

17. A loading/threading mechanism as claimed in Claim 1, wherein the loading mechanism includes a loading mechanism portion and the threading mechanism includes a threading mechanism portion in which the loading mechanism portion has a loader drive gear (6), a loader drive cam groove (6-1) including a circumferential direction groove portion (6-1b) and a non-circumferential direction groove portion (6-1 a) in the loader drive gear, the loader drive cam groove being for engagement with a loader drive engaging member affixed to a loader drive plate, the loader drive plate being engaged with the cartridge tray (2) whereby in the condition in which the non-circumferential direction groove portion of the loader drive cam groove is engaged with the loader drive engaging member, during the rotation of the loader drive gear, the cartridge tray is moved between the eject position and the mount position and in which the threading mechanism portion has a threader drive gear (11) engageable with the loader drive gear, a threader drive cam groove (11-1) including a circumferential direction groove portion (11-1b) and a non-circumferential direction groove portion (11-1a) in the threader drive gear, the threader drive cam groove being engaged with a threader drive engaging member which is affixed to a threader drive arm member rotatable around the rotating centre parallel to the rotational centre of the threading drive gear so as to be eccentric to the rotating centre, the threader drive arm member being connected to the leader block (9), whereby in the condition in which the non-circumferential direction groove portion of the threader drive cam groove is engaged with the threader drive engaging member, during the rotation of the threader drive gear, the leader block is moved through the predetermined route between the unload position and the load position, the loader drive gear and the threader drive gear being engaged with each other, 55

so that, when the loader drive engaging member is engaged with the non-circumferential direction groove portion of the loader drive cam groove, the threader drive engaging member is engaged with the circumferential direction groove portion of the threader drive cam groove, and when the threader drive engaging member is engaged with the non-circumferential direction groove portion of the threader drive cam groove, the loader drive engaging member is engaged with the circumferential direction groove portion of the loader drive cam groove, and the single driving force generating source is a driving motor for driving the rotation of the loader drive gear or the threader drive gear.

18. A loading/threading mechanism as claimed in Claim 1, wherein the loading mechanism includes a loader drive cam groove (6-1) in a driving rotator and the threading mechanism includes a threader drive cam groove (11-1) in the driving rotator, the loader drive cam groove including a first circumferential direction groove portion (6-1 b) and a first non-circumferential direction groove portion (6-1 a) and a threader drive cam groove including a second circumferential direction groove portion (11-1b) and a second non-circumferential direction groove portion (11-1a), the loader drive cam groove is engaged with a loader drive engaging member affixed to a loader drive plate (5), the loader drive plate being engaged with the cartridge tray (2), whereby in the condition in which the first non-circumferential direction groove portion of the loader drive cam groove is engaged with the loader drive engaging member, during the rotation of the rotator, the cartridge tray is moved between the eject position and the mount position, the threader drive cam groove being engaged with a threader drive engaging member which is affixed to a threader drive arm member rotatable around the rotating centre parallel to the rotational centre of the rotator so as to be eccentric to the rotating centre, the threader drive arm being connected to the leader block (9), whereby in the condition in which the second non-circumferential direction groove portion of the threader drive cam groove is engaged with the threader drive engaging member, during the rotation of the rotator, the leader block is moved through the predetermined route between the unload position and the load position, and the loader drive cam groove and the threader drive cam groove having such a phase that, when the loader drive engaging member is engaged with the first non-circumferential direction groove portion of the loader drive cam groove, the threader drive engaging member is engaged with the second circumferential direction groove portion of the threader drive cam groove, and when the threader drive engaging member is engaged with the second non-circumferential direction groove portion of the threader drive cam groove, the

loader drive engaging member is engaged with the first circumferential direction groove portion of the loader drive cam groove.

5 19. A loading/threading mechanism as claimed in either Claim 17 or Claim 18, including a retractor block (12) engageable with the leader block (9) located at the unload position, wherein the retractor block is rotatable between a first attitude and a second attitude and biased so as to assume the first attitude as a result of the biasing means, and the retractor block being engaged with the leader block so that when in the first attitude the leader block is retracted from the cartridge accommodated in the cartridge tray (2) located at the mount position, while in the second attitude the leader block is advanced to the cartridge accommodated in the cartridge tray located at the mount position, the retractor block being set to the second attitude against the biasing force of the biasing means when a retractor cam affixed to the loader drive plate abuts against the retractor block.

10 20. A loading/threading mechanism as claimed in Claim 19, wherein the biasing means is a spring.

15 21. A loading/threading mechanism as claimed in Claim 19, wherein the leader block (9) is rotatably secured to the threader drive arm member and has an engaging projection which is engageable with the retractor block (12) and the retractor block has an engaging groove which is engageable with the engaging projection.

20 22. A loading/threading mechanism as claimed in either Claim 17 or Claim 18, wherein a leader pin capture groove (9-2) engageable with the leader pin (1-7) is formed in the leader block (9).

25 23. A loading/threading mechanism as claimed in either Claim 17 or Claim 18, including a threading guide groove for setting the predetermined route of the movement of the leader block (9), and the threader drive arm member has a threading guide engaging member which is engageable with the threading guide groove.

25 24. A loading/threading mechanism as claimed in either Claim 17 or Claim 18, wherein the threader drive arm member has a first arm portion which is rotatable around the rotating centre and has the threader drive engaging member, and a second arm portion which is rotatably connected to the first arm portion and connected to the leader block (9).

#### Patentansprüche

1. Lade-/Einfädelungsmechanismus für eine Einspu-

lenkassetten- (Single Reel Cartridge) Magnetbandvorrichtung, mit:

einem Lademechanismus (4-6) zum Bewegen eines Kassettenhalters (2) zum Aufnehmen einer Kassette (1) zwischen einer Ausstoßposition, in der die Kassette in den Kassettenhalter ladbar bzw. vom Kassettenhalter entnehmbar ist, und einer Betriebsposition, in der eine Spule (1-2) der Kassette mit einer Kassettenpulendrehrichtung (21) der Magnetbandvorrichtung in Eingriff steht; und einem Einfädelungsmechanismus (7-13, 26, 27, 33) zum Bewegen des Endabschnitts eines in der Kassette aufgenommenen Magnetbandes (1-6) zwischen einer Position im an der Betriebsposition angeordneten Kassettenhalter und einer Position in einer Spule (31) der Magnetbandvorrichtung, wobei der Lademechanismus und der Einfädelungsmechanismus durch eine einzelne Antriebskrafterzeugungsquelle (22, 23) angetrieben werden;

**dadurch gekennzeichnet, dass**

der Einfädelungsmechanismus einen Führungsblock (9) aufweist, der mit einem am Endabschnitt des Magnetbandes befestigten Führungsstift (1-7) lösbar in Eingriff gebracht werden kann, wobei der Einfädelungsmechanismus dazu geeignet ist:

den Führungsblock über eine vorgegebene Strecke zwischen einer Entnahmeposition in der Nähe des an der Betriebsposition angeordneten Kassettenhalters und einer Ladeposition in einer Nut (31-2) des Kernabschnitts der Spule der Magnetbandvorrichtung zu bewegen; und der Führungsblock in der Entnahmeposition entweder an einer zurückgezogenen Position angeordnet ist, an der der Führungsblock nicht mit dem Führungsstift in Eingriff steht und von der Kassette zurückgezogen ist, die in dem an der Betriebsposition angeordneten Kassettenhalter aufgenommen ist, oder an einer Führungsstifteingriffsposition, an der der Führungsblock mit dem Führungsstift in Eingriff steht, der am Endabschnitt des Magnetbandes in der Kassette befestigt ist, die in dem an der Betriebsposition angeordneten Kassettenhalter aufgenommen ist.

2. Lade-/Einfädelungsmechanismus nach Anspruch 1, wobei der Lademechanismus ein Ladeantriebsdrehelement mit einer Ladeantriebsführungsnu (6-1) und einem Ladeantriebselement aufweist, das in eine Richtung hin- und hergehend beweglich ist, um den Kassettenhalter (2) zwischen der

Ausstoßposition und der Betriebsposition zu bewegen, wobei das Ladeantriebselement ein Ladeantriebseingriffselement aufweist, das mit der Ladeantriebsführungsnu in Eingriff steht, wobei der Einfädelungsmechanismus ein Einfädelungsantriebsdrehelement mit einer Einfädelungsantriebsführungsnu (11-1) und ein Einfädelungsantriebselement zum Antreiben des Führungsblocks (9) für eine Bewegung zwischen der Entnahmeposition und der Ladeposition aufweist, wobei das Einfädelungsantriebselement ein Einfädelungsantriebseingriffselement aufweist, das dazu geeignet ist, mit der Einfädelungsantriebsführungsnu in Eingriff zu kommen, und wobei das Ladeantriebsdrehelement und das Einfädelungsantriebsdrehelement miteinander verbunden sind, so dass eine Drehantriebskraft dazwischen übertragen wird, und wobei eine Antriebskraft von der Antriebskrafterzeugungsquelle zum Ladeantriebsdrehelement oder zum Einfädelungsantriebsdrehelement übertragen wird.

3. Lade-/Einfädelungsmechanismus nach Anspruch 1, wobei der Lademechanismus eine in einem Antriebsdrehelement ausgebildete Ladeantriebsführungsnu (6-1) und ein Ladeantriebselement aufweist, das in eine Richtung hin- und hergehend beweglich ist, um den Kassettenhalter (2) zwischen der Ausstoßposition und der Betriebsposition zu bewegen, wobei das Ladeantriebselement ein Ladeantriebseingriffselement aufweist, das dazu geeignet ist, mit der Ladeantriebsführungsnu in Eingriff zu kommen, wobei der Einfädelungsmechanismus eine Einfädelungsantriebsführungsnu (11-1) im Antriebsdrehelement und ein Einfädelungsantriebselement zum Antreiben des Führungsblocks (9) für eine Bewegung zwischen der Entnahmeposition und der Ladeposition aufweist, und wobei das Einfädelungsantriebselement ein Einfädelungsantriebseingriffselement aufweist, das dazu geeignet ist, mit der Einfädelungsantriebsführungsnu in Eingriff zu kommen, so dass eine Antriebskraft von der Antriebskrafterzeugungsquelle zum Antriebsdrehelement übertragen werden kann.

4. Lade-/Einfädelungsmechanismus nach Anspruch 2 oder 3, wobei der Einfädelungsmechanismus einen Retraktor (12) zum Bewegen des Ladeantriebselement in den Zustand aufweist, in dem der Kassettenhalter (2) in der Betriebsposition angeordnet ist, wodurch der Führungsblock (9) von der zurückgezogenen Position zur Führungsstifteingriffsposition bewegt wird, und wobei der Retraktor einen Stift (12-2) aufweist, der dazu geeignet ist, mit einer am Ladeantriebselement befestigten Nocke (5-5) des Retraktors in Eingriff zu kommen, und eine Nut (12-3), die dazu geeignet ist, mit einem am Führungsblock befestigten Stift (9-3) in Eingriff zu kommen.

5. Lade-/Einfädelungsmechanismus nach Anspruch 1, wobei der Lademechanismus und der Einfädelungsmechanismus eine Ladeantriebsführungsnu (6-1) bzw. eine Einfädelungsantriebsführungsnu (11-1) auf einem Drehelement aufweisen, wobei die Ladeantriebsführungsnu einen ersten Nutenabschnitt (6-1b), der sich in der Umfangsrichtung erstreckt, so dass er von der Drehmitte des Drehelements in einem festen Abstand beabstandet gehalten wird, und einen zweiten Nutenabschnitt (6-1a) aufweist, der mit einem Endabschnitt des ersten Nutenabschnitts verbunden ist und sich derart erstreckt, dass der Abstand von der Drehmitte des Drehelements sich ändert, wobei die Ladeantriebsführungsnu mit einem am Ladeantriebselement befestigten Ladeantriebseingriffselement in Eingriff steht und das Ladeantriebselement mit dem Kassettenhalter (2) in Eingriff steht, wodurch in dem Zustand, in dem der zweite Nutenabschnitt mit dem Ladeantriebseingriffselement in Eingriff steht, der Kassettenhalter während der Drehbewegung des Drehelements zwischen der Ausstoßposition und der Betriebspotition bewegt wird, wobei die Einfädelungsantriebsführungsnu einen dritten Nutenabschnitt (11-1b), der sich in der Umfangsrichtung derart erstreckt, dass er in einem festen Abstand von der Drehmitte des Drehelements beabstandet gehalten wird, und einen vierten Nutenabschnitt (11-1a) aufweist, der mit einem Endabschnitt des dritten Nutenabschnitts verbunden ist und sich derart erstreckt, dass der Abstand von der Drehmitte des Drehelements sich ändert, wobei die Einfädelungsantriebsführungsnu mit einem am Einfädelungsantriebselement befestigten Einfädelungsantriebseingriffselement in Eingriff steht und das Einfädelungsantriebseingriffselement mit dem Führungsblock (9) verbunden ist, wodurch in dem Zustand, in dem der vierte Nutenabschnitt mit dem Einfädelungsantriebseingriffselement in Eingriff steht, der Führungsblock während der Drehbewegung des Drehelements über die vorgegebene Strecke bewegt wird, wobei die ersten bis vierten Nutenabschnitte derart konfiguriert sind, dass, wenn das Ladeantriebseingriffselement mit dem zweiten Nutenabschnitt in Eingriff steht, das Einfädelungsantriebseingriffselement mit dem dritten Nutenabschnitt in Eingriff steht, und wenn das Einfädelungsantriebseingriffselement mit dem vierten Nutenabschnitt in Eingriff steht, das Ladeantriebseingriffselement mit dem ersten Nutenabschnitt in Eingriff steht, und wobei die einzelne Antriebskrafterzeugungsquelle die Drehkraft zum Drehelement überträgt.

6. Lade-/Einfädelungsmechanismus nach Anspruch 5, wobei das Drehelement ein erstes Drehelement und ein zweites Drehelement aufweist, die miteinander verbunden sind, so dass sie sich synchron miteinander drehen, wobei die Ladeantriebsführungsnu (6-1) im ersten Drehelement angeordnet ist und die Einfädelungsantriebsführungsnu (11-1) im zweiten Drehelement angeordnet ist.

7. Lade-/Einfädelungsmechanismus nach Anspruch 6, wobei das erste Drehelement ein Ladeantriebszahnrad (6) und das zweite Drehelement ein Einfädelungsantriebszahnrad (11) ist, wobei das Ladeantriebszahnrad und das Einfädelungsantriebszahnrad miteinander in Eingriff stehen.

5 10 15 20 25 30 35 40 45 50 55

8. Lade-/Einfädelungsmechanismus nach Anspruch 5, wobei das Ladeantriebselement den Kassettenhalter (2) bewegt, während ein am Kassettenhalter befestigtes Führungseingriffselement in Eingriff mit einer in einem Ladeführungselement ausgebildeten Ladeführungsnu (4-1, 4-2) geführt wird.

9. Lade-/Einfädelungsmechanismus nach Anspruch 8, wobei die Ladeführungsnu eine Krümmung aufweist.

10. Lade-/Einfädelungsmechanismus nach einem der Ansprüche 5 bis 9, wobei der Führungsblock (9) in der Entnahmeposition an der zurückgezogenen Position oder an der Führungstifteingriffsposition angeordnet ist.

11. Lade-/Einfädelungsmechanismus nach Anspruch 10 mit einem drehbaren Retraktorblock (12) zum Drehen des an der Entnahmeposition angeordneten Führungsblocks (9), wobei der Retraktorblock in eine Drehrichtung vorgespannt ist, wobei das Ladeantriebselement eine Retraktornocke (5-5) aufweist, die mit dem Retraktorblock in Eingriff steht, um den Retraktorblock in die andere Drehrichtung zu drehen, und wobei das Ladeantriebselement bewegt wird, während der Kassettenhalter (2) an der Betriebspotition angeordnet ist, so dass der Führungsblock gegen die Vorspannkraft gedreht und von der zurückgezogenen Position zur Führungsstifteingriffsposition bewegt wird.

12. Lade-/Einfädelungsmechanismus nach Anspruch 11 mit einer Führungsnocke (33) zum Begrenzen des Winkelbereichs der Drehbewegung des Retraktorblocks an der Entnahmeposition.

13. Lade-/Einfädelungsmechanismus nach Anspruch 5, wobei der Kassettenhalter (2) ein Eingriffselement (2-4) aufweist, das dazu geeignet ist, mit einer Tür (1-3) der an der Ausstoßposition angeordneten Kassette in Eingriff zu kommen, um die Tür zu öffnen.

14. Lade-/Einfädelungsmechanismus nach Anspruch 5, wobei der Kassettenhalter (2) ein Verriegelungselement aufweist, das mit einer Verriegelungsöffnung (1-9) der an die Ausstossposition angeordneten Kassette in Eingriff kommt, um die Kassette im Kas-

settenhalter zu fixieren.

15. Lade-/Einfädelungsmechanismus nach Anspruch 5 mit einem Verriegelungsmechanismus zum Fixieren des Kassettenhalters (2) an der Ausstoßposition, wobei der Verriegelungsmechanismus an der schrägen Fläche der Kassette anstößt, um die Fixierung zu lösen. 5

16. Lade-/Einfädelungsmechanismus nach Anspruch 5, wobei der Führungsblock (9) durch einen am Einfädelungsantriebselement befestigten Haltestift mit einem Abstand drehbar gehalten wird und eine Einfädelungsführungsnu zum Führen des Haltestifts aufweist, wenn der Führungsblock zwischen der Entnahmeposition und der Ladeposition bewegt wird. 10

17. Lade-/Einfädelungsmechanismus nach Anspruch 1, wobei der Lademechanismus einen Lademechanismusabschnitt aufweist und der Einfädelungsmechanismus einen Einfädelungsmechanismusabschnitt aufweist, wobei der Lademechanismusabschnitt ein Ladeantriebszahnrad (6), eine Ladeantriebsführungsnu (6-1) mit einem sich in Umfangsrichtung erstreckenden Nutenabschnitt (6-1b) und einem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt (6-1a) im Ladeantriebszahnrad aufweist, wobei die Ladeantriebsführungsnu dazu geeignet ist, mit einem an einer Ladeantriebsplatte befestigten Ladeantriebseingriffselement in Eingriff zu kommen, wobei die Ladeantriebsplatte mit dem Kassettenhalter (2) in Eingriff steht, so dass in dem Zustand, in dem der sich nicht in Umfangsrichtung erstreckende Nutenabschnitt der Ladeantriebsführungsnu mit dem Ladeantriebseingriffselement in Eingriff steht, der Kassettenhalter während der Drehbewegung des Ladeantriebszahnades zwischen der Ausstoßposition und der Betriebsposition bewegt wird, und wobei der Einfädelungsmechanismusabschnitt ein Einfädelungsantriebszahnrad (11), das dazu geeignet ist, mit dem Ladeantriebszahnrad in Eingriff zu kommen, und eine Einfädelungsantriebsführungsnu (11-1) mit einem sich in Umfangsrichtung erstreckenden Nutenabschnitt (11-1b) und einem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt (11-1a) im Einfädelungsantriebszahnrad aufweist, wobei die Einfädelungsantriebsführungsnu mit einem Einfädelungsantriebseingriffselement in Eingriff steht, das an einem Einfädelungsantriebsarmelement befestigt ist, das um die Drehmitte parallel zur Drehmitte des Einfädelungsantriebszahnades drehbar ist, wobei das Einfädelungsantriebsarmelement mit dem Führungsblock (9) verbunden ist, so dass in dem Zustand, in dem der sich nicht in Umfangsrichtung erstreckende Nutenabschnitt der Einfädelungsantriebsführungsnu mit dem Einfädelungsantriebseingriffselement in Eingriff steht, der Führungsblock während der Drehbewegung des Einfädelungsantriebszahnades über eine vorgegebene Strecke zwischen der Entnahmeposition und der Ladeposition bewegt wird, wobei das Ladeantriebszahnrad und das Einfädelungsantriebszahnrad miteinander in Eingriff stehen, so dass, wenn das Ladeantriebseingriffselement mit dem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsführungsnu in Eingriff steht, das Einfädelungsantriebseingriffselement mit dem sich in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu in Eingriff steht, und wenn das Einfädelungsantriebseingriffselement mit dem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu in Eingriff steht, das Ladeantriebseingriffselement mit dem sich in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsführungsnu in Eingriff steht, und wobei die einzelne Antriebskrafterzeugungsquelle ein Antriebsmotor zum Erzeugen der Drehbewegung des Ladeantriebszahnades oder des Einfädelungsantriebszahnades ist. 15

18. Lade-/Einfädelungsmechanismus nach Anspruch 1, wobei der Lademechanismus eine Ladeantriebsführungsnu (6-1) in einem Antriebsdrehelement aufweist und der Einfädelungsmechanismus eine Einfädelungsantriebsführungsnu (11-1) im Antriebsdrehelement aufweist, wobei die Ladeantriebsführungsnu einen ersten sich in Umfangsrichtung erstreckenden Nutenabschnitt (6-1b) und einen ersten sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt (6-1a) und eine Einfädelungsantriebsführungsnu mit einem zweiten sich in Umfangsrichtung erstreckenden Nutenabschnitt (11-1b) und einem zweiten sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt (11-1a) aufweist, wobei die Ladeantriebsführungsnu mit einem an einer Ladeantriebsplatte (5) befestigten Ladeantriebseingriffselement in Eingriff steht, wobei die Ladeantriebsplatte mit dem Kassettenhalter (2) in Eingriff steht, so dass in dem Zustand, in dem der erste sich nicht in Umfangsrichtung erstreckende Nutenabschnitt der Ladeantriebsführungsnu mit dem Ladeantriebseingriffselement in Eingriff steht, der Kassettenhalter während der Drehbewegung des Drehelements zwischen der Ausstoßposition und der Betriebsposition bewegt wird, die Einfädelungsantriebsführungsnu mit einem Einfädelungsantriebseingriffselement in Eingriff steht, das an einem Einfädelungsantriebsarmelement befestigt ist, das um die Drehmitte parallel zur Drehmitte des Drehelements drehbar ist, so dass es exzentrisch zur Drehmitte angeordnet ist, wobei das Einfädelungsantriebsarmelement mit dem Führungsblock (9) verbunden ist, so dass in dem Zustand, in dem der zweite sich nicht in Umfangsrichtung erstreckende Nutenabschnitt der Einfädelungsantriebsführungsnu mit dem Einfädelungsantriebseingriffselement in Eingriff steht, der Führungsblock während der Drehbewegung des Drehelements über eine vorgegebene Strecke zwischen der Entnahmeposition und der Ladeposition bewegt wird, wobei das Ladeantriebszahnrad und das Einfädelungsantriebszahnrad miteinander in Eingriff stehen, so dass, wenn das Ladeantriebseingriffselement mit dem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsführungsnu in Eingriff steht, das Einfädelungsantriebseingriffselement mit dem sich in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu in Eingriff steht, und wenn das Einfädelungsantriebseingriffselement mit dem sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu in Eingriff steht, das Ladeantriebseingriffselement mit dem sich in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsführungsnu in Eingriff steht, und wobei die einzelne Antriebskrafterzeugungsquelle ein Antriebsmotor zum Erzeugen der Drehbewegung des Ladeantriebszahnades oder des Einfädelungsantriebszahnades ist. 20

25

30

35

40

45

50

55

triebseingriffselement in Eingriff steht, der Führungsblock während der Drehbewegung des Drehelements über eine vorgegebene Strecke zwischen der Entnahmeposition und der Ladeposition bewegt wird, wobei die Ladeantriebsführungsnu 10 und die Einfädelungsantriebsführungsnu 15 eine derartige Phase haben, dass, wenn das Ladeantriebseingriffselement mit dem ersten sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsantriebsführungsnu 20 in Eingriff steht, das Führungsantriebseingriffselement mit dem zweiten sich in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu 25 in Eingriff steht, und wenn das Einfädelungsantriebseingriffselement mit dem zweiten sich nicht in Umfangsrichtung erstreckenden Nutenabschnitt der Einfädelungsantriebsführungsnu 30 in Eingriff steht, das Ladeantriebseingriffselement mit dem ersten sich in Umfangsrichtung erstreckenden Nutenabschnitt der Ladeantriebsführungsnu 35 in Eingriff steht.

19. Lade-/Einfädelungsmechanismus nach Anspruch 17 oder 18 mit einem Retraktorblock (12), der dazu geeignet ist, mit dem an der Entnahmeposition angeordneten Führungsblock (9) in Eingriff zu kommen, wobei der Retraktorblock zwischen einer ersten Position und einer zweiten Position drehbar und durch eine Vorspannungseinrichtung auf die erste Position vorgespannt ist, wobei der Retraktorblock mit dem Führungsblock in Eingriff steht, so dass, wenn der Führungsblock auf die erste Position eingestellt ist, der Führungsblock von der Kassette zurückgezogen ist, die in dem an der Betriebsp 40 position angeordneten Kassettenhalter (2) aufgenommen ist, während der Führungsblock in der zweiten Position zur Kassette bewegt wird, die in dem in der Betriebsp 45 position angeordneten Kassettenhalter aufgenommen ist, wobei der Retraktorblock gegen die Vorspannkraft der Vorspannungseinrichtung auf die zweite Position eingestellt wird, wenn eine an der Ladeantriebsplatte befestigte Retraktornocke am Retraktorblock anstößt.

20. Lade-/Einfädelungsmechanismus nach Anspruch 19, wobei die Vorspannungseinrichtung eine Feder ist. 45

21. Lade-/Einfädelungsmechanismus nach Anspruch 19, wobei der Führungsblock (9) am Einfädelungsantriebsarmelement drehbar befestigt ist und einen Eingriffsvorprung aufweist, der dazu geeignet ist, mit dem Retraktorblock (12) in Eingriff zu kommen, und wobei der Retraktorblock eine Eingriffsnut aufweist, die dazu geeignet ist, mit dem Eingriffsvorprung in Eingriff zu kommen. 50

22. Lade-/Einfädelungsmechanismus nach Anspruch 17 oder 18, wobei eine Führungsstiftaufnahmenut 55

(9-2), die dazu geeignet ist, mit dem Führungsstift (1-7) in Eingriff zu kommen, im Führungsblock (9) ausgebildet ist.

5 23. Lade-/Einfädelungsmechanismus nach Anspruch 17 oder 18 mit einer Einfädelungsführungsnu 10 zum Einstellen der vorgegebenen Bewegungsstrecke des Führungsblocks (9), wobei das Einfädelungsantriebsarmelement ein Einfädelungsführungsnu 15 eingriffselement aufweist, das dazu geeignet ist, mit der Einfädelungsführungsnu 20 in Eingriff zu kommen.

24. Lade-/Einfädelungsmechanismus nach Anspruch 17 oder 18, wobei das Einfädelungsantriebsarmelement einen ersten Armabschnitt, der um die Drehmitte drehbar ist und das Einfädelungsantriebseingriffselement aufweist, und einen zweiten Armabschnitt aufweist, der mit dem ersten Armabschnitt drehbar verbunden ist und mit dem Führungsblock (9) verbunden ist. 25

### Revendications

25. 1. Mécanisme de chargement/d'embobinage d'un appareil à bande magnétique doté d'une seule cartouche de bobine, comportant un mécanisme de chargement (4-6) destiné à déplacer un plateau pour cartouche (2) destiné à loger une cartouche (1) entre une position d'éjection dans laquelle la cartouche peut être montée à l'intérieur du plateau pour cartouche/démontée de celui-ci et une position de montage dans laquelle une bobine (1-2) de la cartouche est mise en prise avec des moyens de rotation de bobine de cartouche (21) de l'appareil à bande magnétique, et un mécanisme d'embobinage (7-13, 26, 27, 33) destiné à déplacer la partie d'extrémité d'une bande magnétique (1-6) logée dans la cartouche entre une position dans le plateau pour cartouche situé dans la position de montage et une position dans une bobine (31) de l'appareil à bande magnétique, le mécanisme de chargement et le mécanisme d'embobinage étant entraînés au moyen d'une source (22, 23) générant une seule force d'entraînement ; et **caractérisé en ce que**

le mécanisme d'embobinage comporte un bloc d'amorce (9) qui peut être mis en prise de manière détachable avec une broche d'amorce (1-7) fixé à la partie d'extrémité de la bande magnétique, le mécanisme d'embobinage étant adapté de telle sorte que

le bloc d'amorce est déplacé le long d'un itinéraire prédéterminé entre une position de déchargement au voisinage du plateau pour cartouche situé dans la position de montage et une position de chargement dans une encoche (31-2) de la partie centrale de la bobine de l'appareil à bande

magnétique ; et dans la position de déchargement, le bloc d'amorce est situé soit dans une position rétractée dans laquelle le bloc d'amorce est dégagé de la broche d'amorce et rétracté par rapport à la cartouche logée dans le plateau pour cartouche situé dans la position de montage, soit dans une position de broche d'amorce capturée dans laquelle le bloc d'amorce est mis en prise avec la broche d'amorce fixée à la partie d'extrémité de la bande magnétique dans la cartouche logée dans le plateau pour cartouche situé dans la position de montage.

2. Mécanisme de chargement/d'embobinage selon la revendication 1, dans lequel le mécanisme de chargement présente un rotateur d'entraînement de chargeur présentant une rainure de came d'entraînement de chargeur (6-1), et un élément d'entraînement de chargeur qui peut être déplacé en va-et-vient dans une direction pour faire translater le plateau pour cartouche (2) entre la position d'éjection et la position de montage, l'élément d'entraînement de chargeur présentant un élément de mise en prise d'entraînement de chargeur qui est mis en prise avec la rainure de came d'entraînement de chargeur, le mécanisme d'embobinage présentant un rotateur d'entraînement de dispositif d'embobinage doté d'une rainure de came d'entraînement de dispositif d'embobinage (11-1) et un élément d'entraînement de dispositif d'embobinage destiné à entraîner le bloc d'amorce (9) pour se déplacer entre la position de déchargement et la position de chargement, l'élément d'entraînement de dispositif d'embobinage présentant un élément de mise en prise d'entraînement de dispositif d'embobinage destiné à se mettre en prise avec la rainure de came d'entraînement de dispositif d'embobinage, et le rotateur d'entraînement de chargeur et le rotateur d'entraînement de dispositif d'embobinage étant raccordés l'un à l'autre, de sorte qu'une force d'entraînement rotationnelle est transmise entre ceux-ci, et une force d'entraînement est transmise de la source générant une force d'entraînement à un du rotateur d'entraînement de chargeur et du rotateur d'entraînement d'embobineur.

3. Mécanisme de chargement/d'embobinage selon la revendication 1, dans lequel le mécanisme de chargement présente une rainure de came d'entraînement de chargeur (6-1) formée dans un rotateur d'entraînement, et un élément d'entraînement de chargeur qui peut être déplacé en va-et-vient dans une direction pour faire translater le plateau pour cartouche (2) entre la position d'éjection et la position de montage, l'élément d'entraînement de chargeur présentant un élément de mise en prise d'entraînement de chargeur destiné à se mettre en prise avec la rainure de came d'entraînement de chargeur (6-1) dans le rotateur d'entraînement et un élément d'entraînement d'embobineur destiné à entraîner le bloc d'amorce (9) pour se déplacer entre la position de déchargement et la position de chargement, et l'élément d'entraînement d'embobineur présentant un élément de mise en prise d'entraînement d'embobineur afin de se mettre en prise avec la rainure de came d'entraînement d'embobineur, une force d'entraînement pouvant être transmise de la source générant la force d'entraînement au rotateur d'entraînement.

4. Mécanisme de chargement/d'embobinage selon la revendication 2 ou la revendication 3, dans lequel le mécanisme d'embobinage présente un rétracteur (12) destiné à déplacer l'élément d'entraînement de chargeur dans l'état dans lequel le plateau pour cartouche (2) se situe dans la position de montage, déplaçant de ce fait le bloc d'amorce (9) de la position rétractée à la position de broche d'amorce capturée, et le rétracteur présentant une broche (12-2) qui peut être mise en prise avec une came de rétracteur (5-5) fixée à l'élément d'entraînement de chargeur, et une rainure (12-3) qui peut être mise en prise avec une broche (9-3) fixée au bloc d'amorce.

5. Mécanisme de chargement/d'embobinage selon la revendication 1, dans lequel le mécanisme de chargement et le mécanisme d'embobinage comportent une rainure de came d'entraînement de chargeur (6-1) et une rainure de came d'entraînement d'embobineur (11-1) sur un élément rotatif respectivement, la rainure de came d'entraînement de chargeur présentant une première partie de rainure (6-1b) s'étendant dans la direction circonférentielle de manière à être maintenue à distance du centre de rotation de l'élément rotatif à une distance fixe, et une deuxième partie de rainure (6-1a) qui est reliée à une partie d'extrémité de la première partie de rainure et s'étend de sorte que la distance au centre de rotation de l'élément rotatif est variable, la rainure de came d'entraînement de chargeur étant mise en prise avec un élément de mise en prise d'entraînement de chargeur fixé à un élément d'entraînement de chargeur, l'élément d'entraînement de chargeur étant mis en prise avec le plateau pour cartouche (2), sachant que dans l'état dans lequel la deuxième partie de rainure est mise en prise avec l'élément de mise en prise d'entraînement de chargeur, pendant la rotation de l'élément rotatif, le plateau pour cartouche est déplacé entre la position d'éjection et la position de montage, la rainure de came d'entraînement d'embobineur présentant une troisième partie de rainure (11-1b) s'étendant dans la direction circonférentielle de manière à être maintenue à distan-

ce du centre de rotation de l'élément rotatif à une distance fixe et une quatrième partie de rainure (11-1a) qui est reliée à une partie d'extrémité de la troisième partie de rainure et s'étend de sorte que la distance partant du centre de rotation de l'élément rotatif est variable, la rainure de came d'entraînement d'embobineur étant mise en prise avec un élément de mise en prise d'entraînement d'embobineur fixé à un élément d'entraînement d'embobineur, l'élément d'entraînement d'embobineur étant raccordé au bloc d'amorce (9), sachant que dans l'état dans lequel la quatrième partie de rainure est mise en prise avec l'élément de mise en prise d'entraînement d'embobineur, pendant la rotation de l'élément rotatif, le bloc d'amorce est déplacé le long de l'itinéraire prédéterminé, les première à quatrième parties de rainure étant placées de sorte que lorsque l'élément de mise en prise d'entraînement de chargeur est mis en prise avec la deuxième partie de rainure, l'élément de mise en prise d'entraînement d'embobineur est mis en prise avec la troisième partie de rainure, et lorsque l'élément de mise en prise d'entraînement de dispositif d'embobinage est mis en prise avec la quatrième partie de rainure, l'élément de mise en prise d'entraînement de chargeur est mis en prise avec la première partie de rainure, et la source générant une seule force d'entraînement transfère la force de rotation à l'élément rotatif.

6. Mécanisme de chargement/d'embobinage selon la revendication 5, dans lequel l'élément rotatif présente une premier rotateur et un second rotateur qui sont raccordés l'un à l'autre de manière à pouvoir tourner de manière synchrone l'un avec l'autre, la rainure de came d'entraînement de chargeur (6-1) se situant dans le premier rotateur, et la rainure de came d'entraînement d'embobineur (11-1) se situant dans le second rotateur.

7. Mécanisme de chargement/d'embobinage selon la revendication 6, dans lequel le premier rotateur est une roue dentée d'entraînement de chargeur (6), et le second rotateur est une roue dentée d'entraînement d'embobineur (11), la roue dentée d'entraînement de chargeur et la roue d'entraînement d'embobineur étant mises en prise l'une avec l'autre.

8. Mécanisme de chargement/d'embobinage selon la revendication 5, dans lequel l'élément d'entraînement de chargeur déplace le plateau pour cartouche (2) tandis qu'un élément de mise en prise de guidage fixé au plateau pour cartouche est guidé en prise avec une rainure de guidage de chargeur (4-1, 4-2) formée dans un élément de guidage de chargeur.

9. Mécanisme de chargement/d'embobinage selon la revendication 8, dans lequel la rainure de guidage de chargeur présente un coude à l'intérieur de celle-

ci.

10. Mécanisme de chargement/d'embobinage selon l'une ou plusieurs des revendications 5 à 9, dans lequel le bloc d'amorce (9) est situé dans une de la position rétractée et de la position de broche d'amorce capturée dans la position de déchargement.

11. Mécanisme de chargement/d'embobinage selon la revendication 10, comportant un bloc d'enrouleur (12) qui peut être amené en rotation pour faire tourner le bloc d'amorce (9) situé dans la position de déchargement, le bloc d'enrouleur étant sollicité dans une direction rotationnelle, l'élément d'entraînement de chargeur présentant une came d'enrouleur (5-5) qui est mise en prise avec le bloc d'enrouleur pour amener le bloc d'enrouleur en rotation dans l'autre direction rotationnelle, et l'élément d'entraînement de chargeur étant déplacé tandis que le plateau pour cartouche (2) est situé dans la position de montage, le bloc d'amorce étant amené en rotation dans le sens inverse de la force de sollicitation pour être déplacé de la position rétractée à la position de broche d'amorce capturée.

12. Mécanisme de chargement/d'embobinage selon la revendication 11, comportant une came de guidage (33) destinée à limiter la plage de rotation angulaire du bloc d'enrouleur dans la position de déchargement.

13. Mécanisme de chargement/d'embobinage selon la revendication 5, dans lequel le plateau pour cartouche (2) présente un élément de mise en prise (2-4) destiné à se mettre en prise avec une porte (1-3) de la cartouche insérée dans la position d'éjection pour ouvrir la porte.

14. Mécanisme de chargement/d'embobinage selon la revendication 5, dans lequel le plateau pour cartouche (2) présente un élément de verrouillage qui est mis en prise avec un trou de verrouillage (1-9) de la cartouche insérée dans la position d'éjection pour fixer la cartouche au plateau pour cartouche.

15. Mécanisme de chargement/d'embobinage selon la revendication 5, comportant un mécanisme de verrouillage destiné à fixer le plateau pour cartouche (2) dans la position d'éjection, le mécanisme de verrouillage étant en butée contre la surface inclinée de la cartouche pour libérer la fixation.

16. Mécanisme de chargement/d'embobinage selon la revendication 5, dans lequel le bloc d'amorce (9) est maintenu de manière rotative avec un jeu par une broche de support fixée à l'élément d'entraînement d'embobineur, et comportant une rainure de came d'embobinage destinée à guider la broche de sup-

port lorsque le bloc d'amorce est déplacé entre la position de déchargement et la position de chargement.

17. Mécanisme de chargement/d'embobinage selon la revendication 1, dans lequel le mécanisme de chargement comporte une partie de mécanisme de chargement et le mécanisme d'embobinage comporte une partie de mécanisme d'embobinage dans laquelle la partie de mécanisme de chargement présente une roue dentée d'entraînement de chargeur (6), une rainure de came d'entraînement de chargeur (6-1) comportant une partie de rainure dans la direction circonférentielle (6-1b) et une partie de rainure de direction non-circonférentielle (6-1a) dans la roue dentée d'entraînement de chargeur, la rainure de came d'entraînement de chargeur étant destinée à se mettre en prise avec un élément de mise en prise d'entraînement de chargeur fixé à une plaque d'entraînement de chargeur, la plaque d'entraînement de chargeur étant mise en prise avec le plateau pour cartouche (2), sachant que dans l'état dans lequel la partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement de chargeur est mise en prise avec l'élément de mise en prise d'entraînement de chargeur, pendant la rotation de la roue dentée d'entraînement de chargeur, le plateau pour cartouche est déplacé entre la position d'éjection et la position de montage et dans laquelle la partie de mécanisme d'embobinage présente une roue dentée d'entraînement d'embobineur (11) pouvant être mis en prise avec une roue dentée d'entraînement de chargeur, une rainure de came d'entraînement d'embobineur (11-1) comportant une partie de rainure dans la direction circonférentielle (11-1b) et une partie de rainure dans la direction non-circonférentielle (11-1a) dans la roue dentée d'entraînement d'embobineur, la rainure de came d'entraînement d'embobineur étant mise en prise avec un élément de mise en prise d'entraînement d'embobineur qui est fixé à un élément de bras d'entraînement d'embobineur pouvant tourner autour du centre de rotation parallèlement au centre de rotation de la roue dentée d'entraînement d'embobineur de manière à être excentrique par rapport au centre de rotation, l'élément de bras d'entraînement d'embobineur étant raccordé au bloc d'amorce (9), sachant que dans l'état dans lequel la partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement d'embobineur est mise en prise avec l'élément de mise en prise d'entraînement d'embobineur, pendant la rotation de la roue dentée d'entraînement d'embobineur, le bloc d'amorce est déplacé le long de l'itinéraire prédéterminé entre la position de déchargement et la position de chargement, la roue dentée d'entraînement de chargeur et la roue dentée d'entraînement d'embobineur étant mises en prise l'une avec l'autre, de

sorte que, lorsque l'élément de mise en prise d'entraînement de chargeur est mis en prise avec la partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement de chargeur, l'élément de mise en prise d'entraînement d'embobineur est mis en prise avec la partie de rainure dans la direction circonférentielle de la rainure de came d'entraînement d'embobineur, et lorsque l'élément de mise en prise d'entraînement d'embobineur est mis en prise avec la partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement d'embobineur, l'élément de mise en prise d'entraînement de chargeur est mis en prise avec la partie de rainure dans la direction circonférentielle de la rainure de came d'entraînement de chargeur, et la source générant une seule force d'entraînement est un moteur d'entraînement destiné à entraîner la rotation de la roue dentée d'entraînement de chargeur ou de la roue dentée d'entraînement d'embobineur.

18. Mécanisme de chargement/d'embobinage selon la revendication 1, dans lequel le mécanisme de chargement comporte une rainure de came d'entraînement de chargeur (6-1) dans un rotateur d'entraînement et le mécanisme d'embobinage comporte une rainure de came d'entraînement d'embobineur (11-1) dans le rotateur d'entraînement, la rainure de came d'entraînement de chargeur comportant une première partie de rainure dans la direction circonférentielle (6-1b) et une première partie de rainure dans la direction non-circonférentielle (6-1a) et une rainure de came d'entraînement d'embobineur comportant une deuxième partie de rainure dans la direction circonférentielle (11-1b) et une deuxième partie de rainure dans la direction non-circonférentielle (11-1a), la rainure de came d'entraînement de chargeur est mise en prise avec un élément de mise en prise d'entraînement de chargeur fixé à une plaque d'entraînement de chargeur (5), la plaque d'entraînement de chargeur étant mise en prise avec le plateau pour cartouche (2), sachant que dans l'état dans lequel la première partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement de chargeur est mise en prise avec l'élément de mise en prise d'entraînement de chargeur, pendant la rotation du rotateur, le plateau pour cartouche est déplacé entre la position d'éjection et la position de montage, la rainure de came d'entraînement d'embobineur étant mise en prise avec un élément de mise en prise d'entraînement d'embobineur qui est fixé à un élément de bras d'entraînement d'embobineur pouvant tourner autour du centre de rotation parallèlement au centre de rotation du rotateur de manière à être excentrique par rapport au centre de rotation, le bras d'entraînement d'embobineur étant raccordé au bloc d'amorce (9), sachant que dans l'état dans lequel la deuxième partie de

rainure dans la direction non-circonférentielle de la rainure de came d'entraînement d'embobineur est mise en prise avec l'élément de mise en prise d'entraînement d'embobineur, pendant la rotation du rotateur, le bloc d'amorce est déplacé le long de l'itinéraire prédéterminé entre la position de déchargement et la position de chargement, et la rainure de came d'entraînement de chargeur et la rainure de came d'entraînement d'embobineur présentant une phase telle que, lorsque l'élément de mise en prise d'entraînement de chargeur est mis en prise avec la première partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement de chargeur, l'élément de mise en prise d'entraînement d'embobineur est mis en prise avec la deuxième partie de rainure dans la direction circonférentielle de la rainure de came d'entraînement d'embobineur, et lorsque l'élément de mise en prise d'entraînement d'embobineur est mis en prise avec la deuxième partie de rainure dans la direction non-circonférentielle de la rainure de came d'entraînement d'embobineur, l'élément de mise en prise de chargeur est mis en prise avec la première partie de rainure dans la direction circonférentielle de la rainure de came d'entraînement de chargeur.

10

19. Mécanisme de chargement/d'embobinage selon la revendication 17 ou la revendication 18, comportant un bloc de rétracteur (12) pouvant être mis en prise avec le bloc d'amorce (9) situé dans la position de déchargement, dans lequel le bloc de rétracteur peut tourner entre une première orientation et une seconde orientation et est sollicité de manière à prendre la première orientation du fait des moyens de sollicitation, et le bloc de rétracteur étant mis en prise avec le bloc d'amorce de sorte que lorsqu'il se trouve dans la première orientation, le bloc d'amorce est rétracté par rapport à la cartouche logée dans le plateau pour cartouche (2) situé dans la position de montage, tandis que dans la seconde orientation, le bloc d'amorce est avancé vers la cartouche logée dans le plateau pour cartouche situé dans la position de montage, le bloc de rétracteur étant placé dans la seconde orientation dans le sens inverse de la force de sollicitation des moyens de sollicitation lorsqu'une came de rétracteur fixée à la plaque d'entraînement de chargeur est en butée contre le bloc de rétracteur.

15

20. Mécanisme de chargement/d'embobinage selon la revendication 19, dans lequel les moyens de sollicitation sont un ressort.

20

21. Mécanisme de chargement/d'embobinage selon la revendication 19, dans lequel le bloc d'amorce (9) est fixé de manière rotative à l'élément de bras d'entraînement d'embobineur et présente une saillie de mise en prise qui peut être mise en prise avec le bloc

25

30

35

40

45

50

55

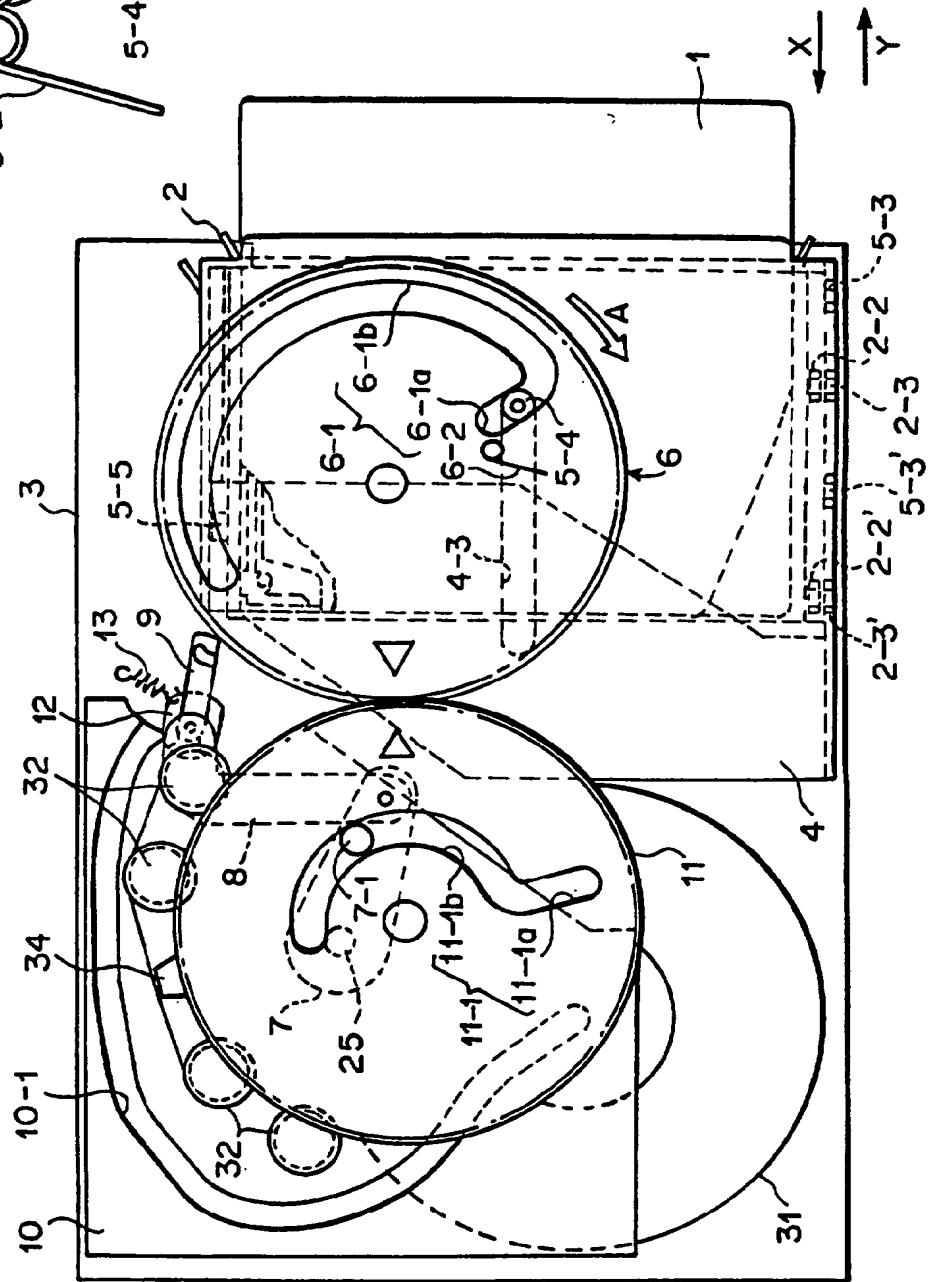
de rétraction (12) et le bloc de rétracteur présente une rainure de mise en prise qui peut être mise en prise avec la saillie de mise en prise.

5 22. Mécanisme de chargement/d'embobinage selon la revendication 17 ou la revendication 18, dans lequel une rainure de capture de broche d'amorce (9-2) pouvant être mise en prise avec la broche d'amorce (1-7) est formée dans le bloc d'amorce (9).

23. Mécanisme de chargement/d'embobinage selon la revendication 17 ou la revendication 18, comportant une rainure de guidage d'embobinage destinée à régler l'itinéraire prédéterminé du déplacement du bloc d'amorce (9), et l'élément de bras d'entraînement d'embobineur présente un élément de mise en prise de guidage d'embobinage qui peut être mis en prise avec la rainure de guidage d'embobinage.

24. Mécanisme de chargement/d'embobinage selon la revendication 17 ou la revendication 18, dans lequel l'élément de bras d'entraînement d'embobineur présente une première partie de bras qui peut tourner autour du centre de rotation et présente l'élément de mise en prise d'entraînement d'embobineur, et une seconde partie de bras qui est raccordée de manière rotative à la première partie de bras et raccordée au bloc d'amorce (9).

FIG. 1A



181

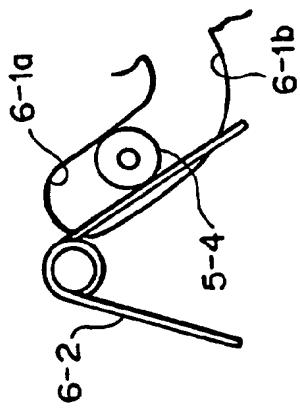


FIG. 2

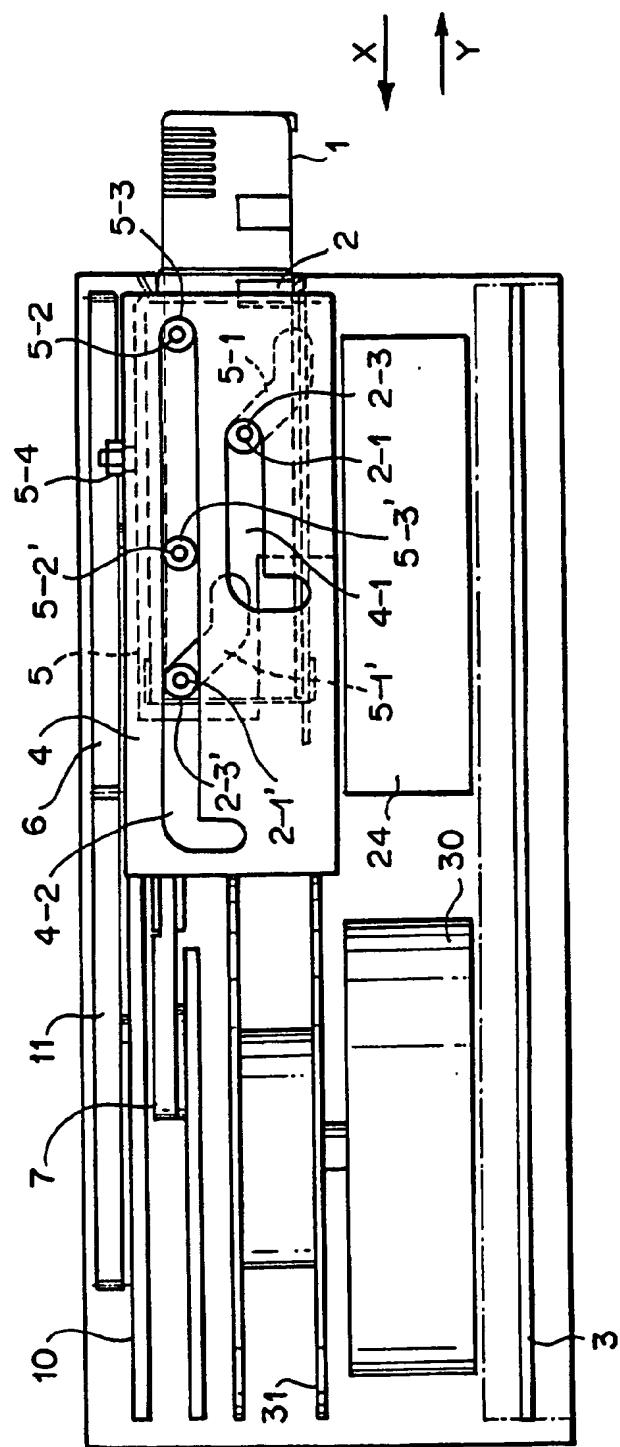


FIG. 3

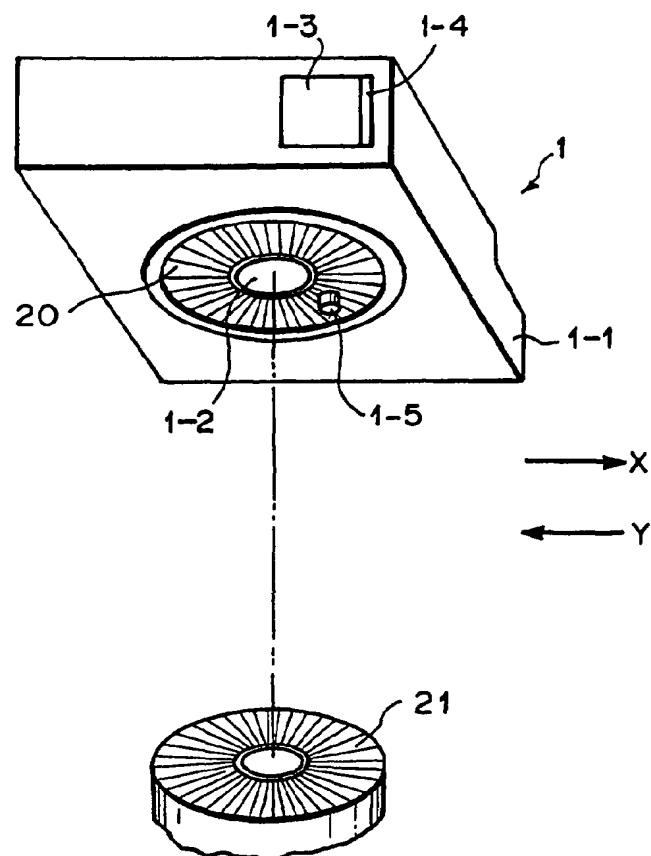


FIG. 4

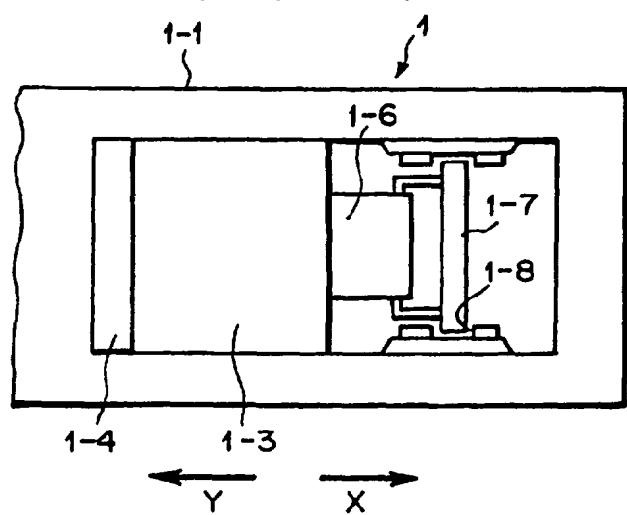


FIG. 5

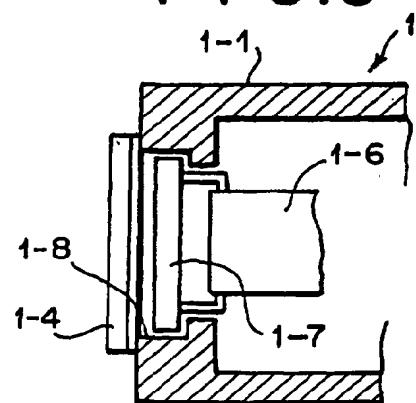


FIG. 6

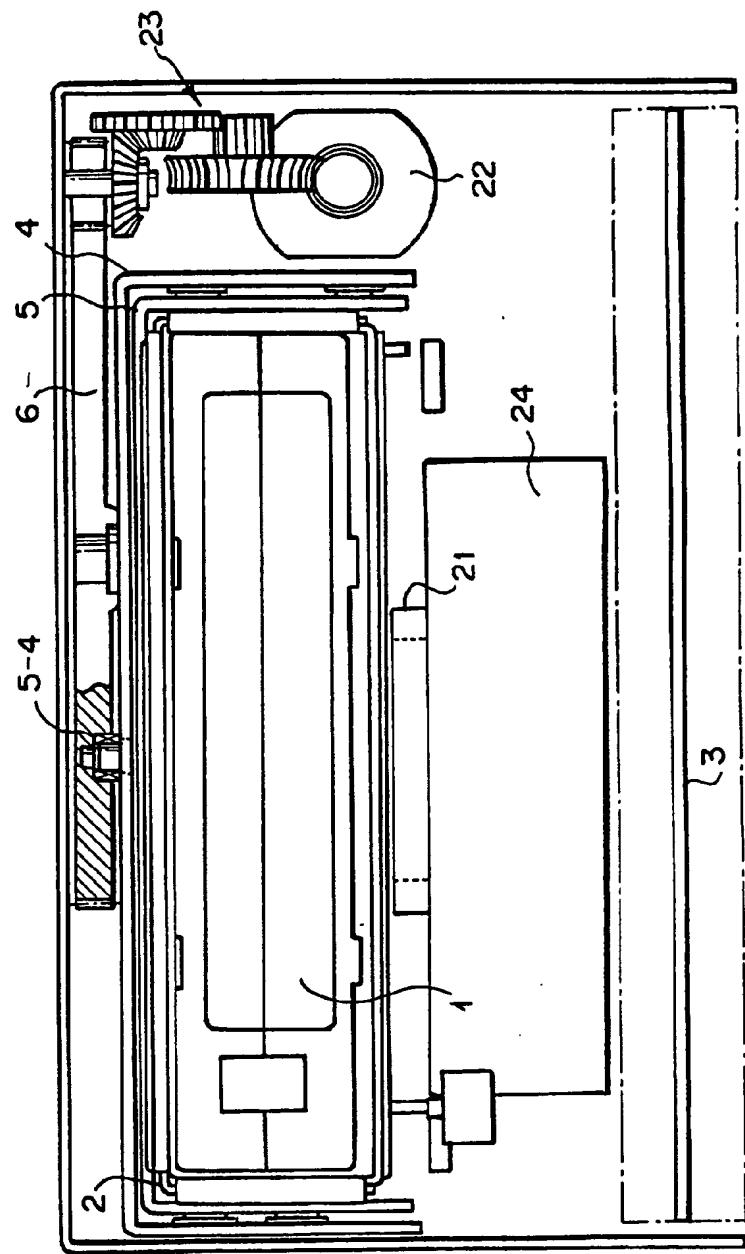
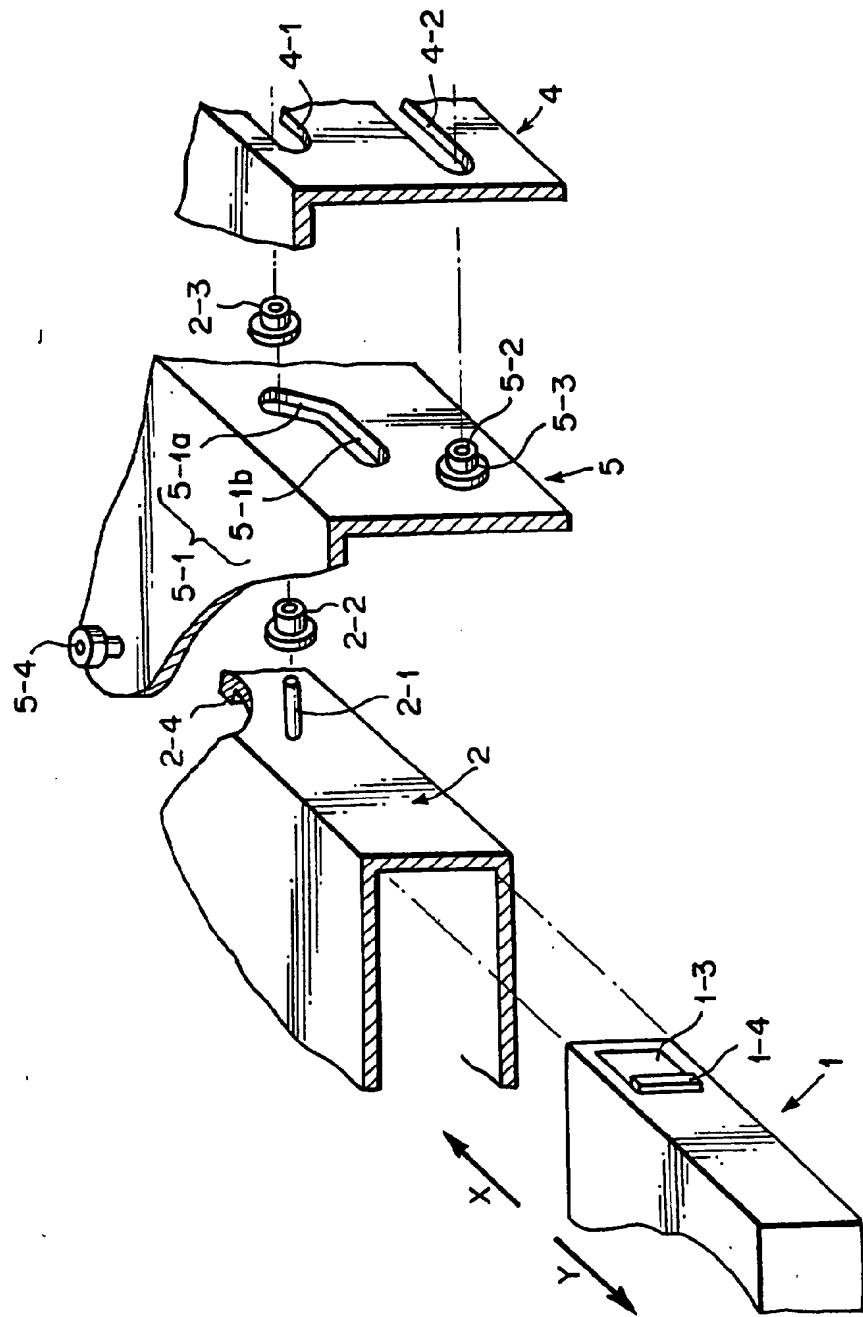
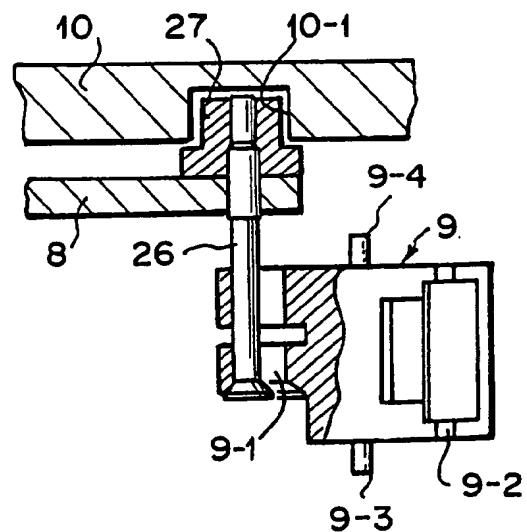


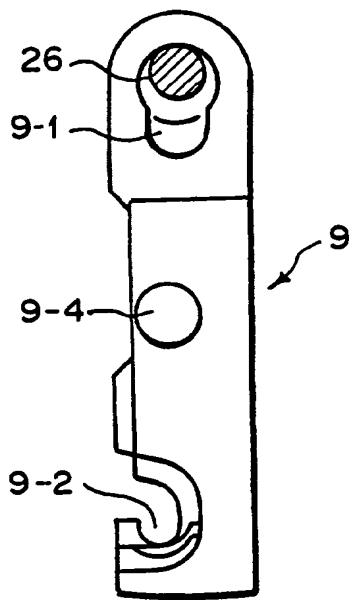
FIG. 7



**F I G.8**



**F I G.9**



F I G.10

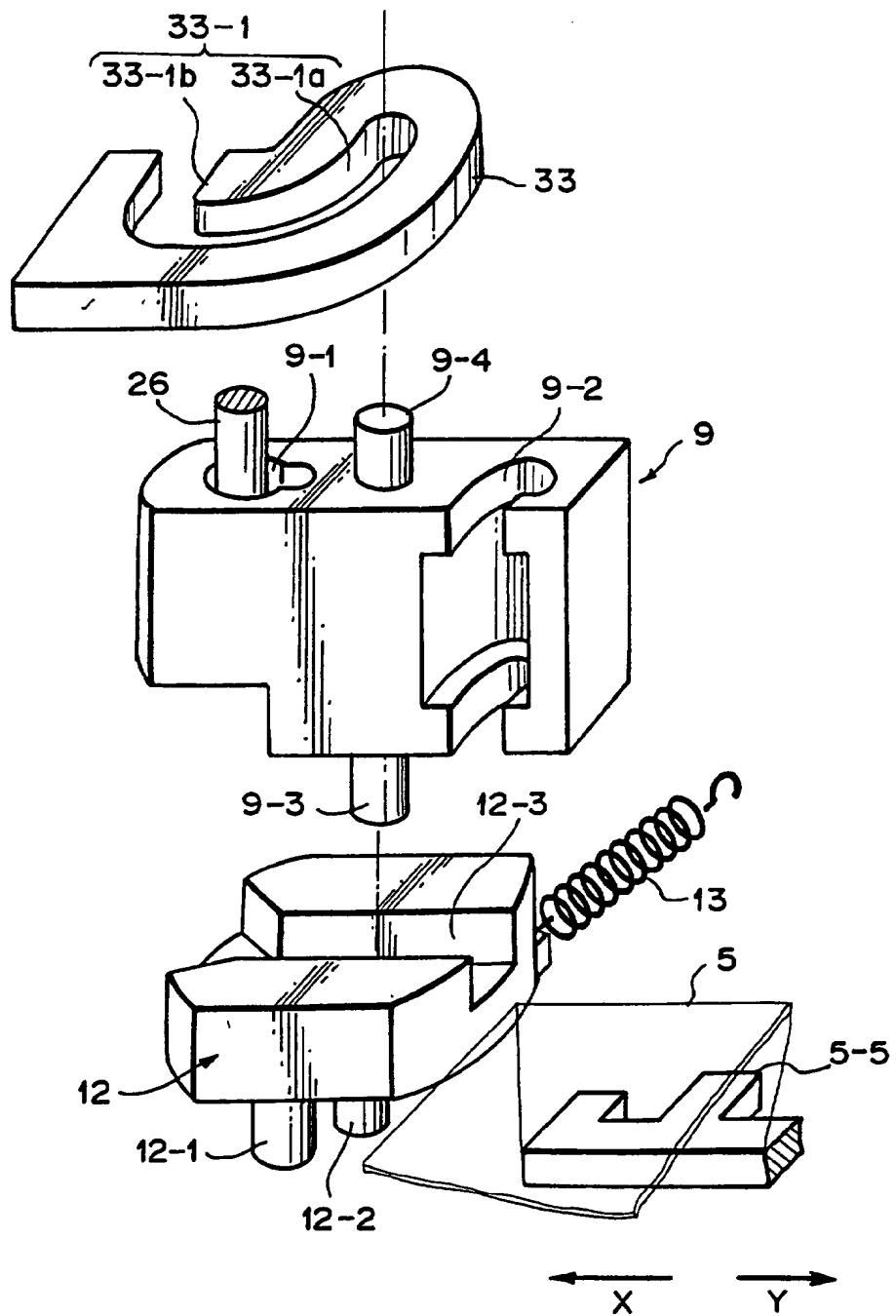
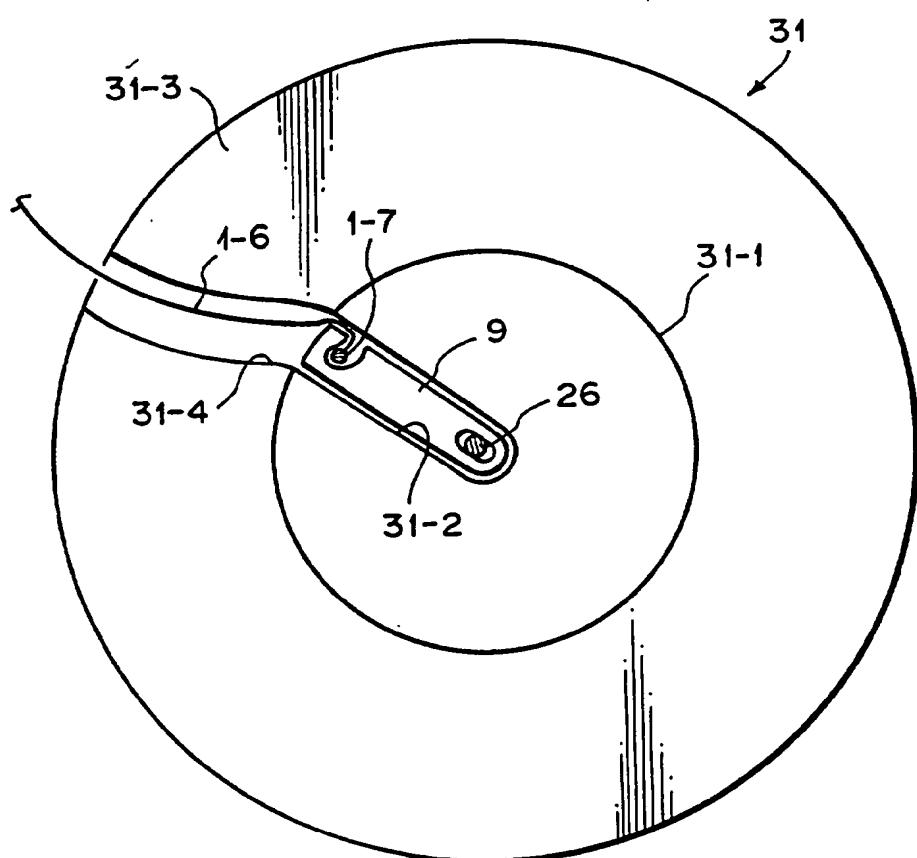
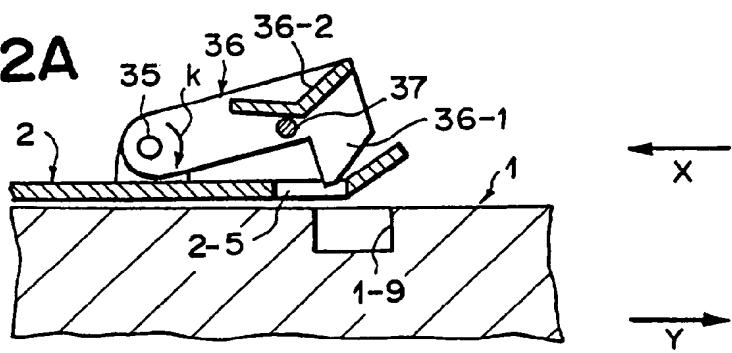


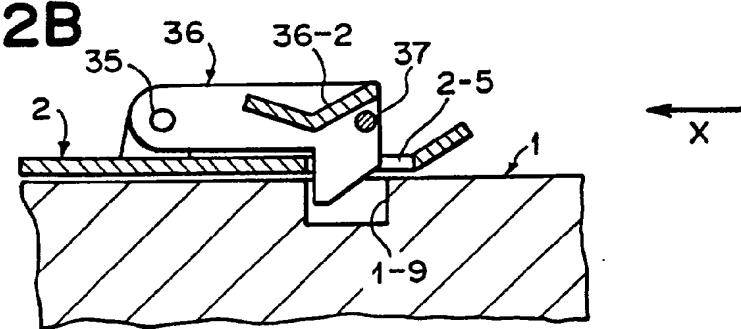
FIG.11



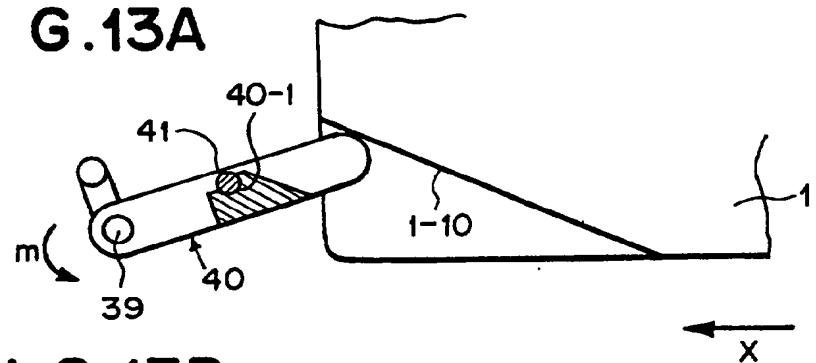
F I G. 12A



F I G. 12B



F I G. 13A



F I G. 13B

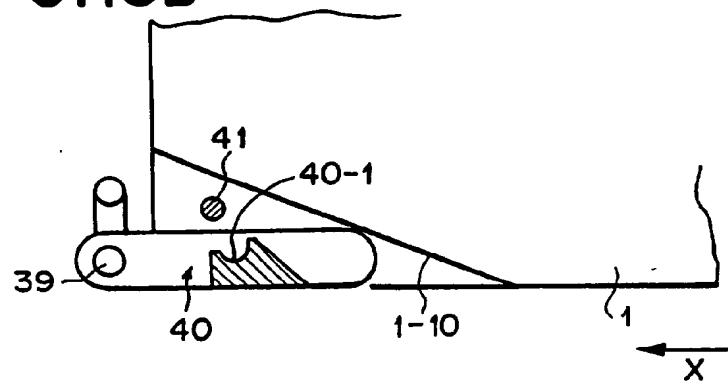


FIG.14

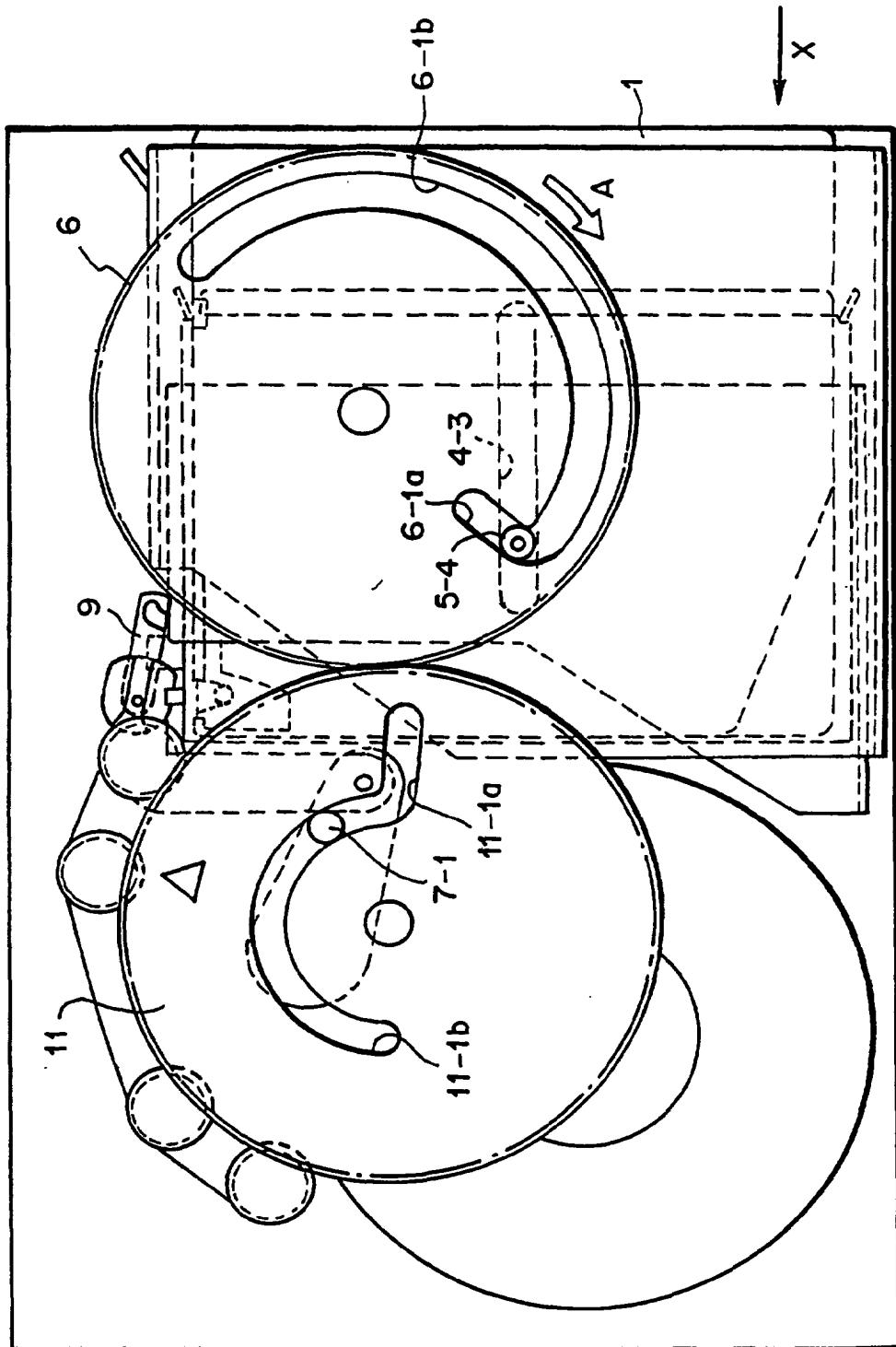


FIG.15

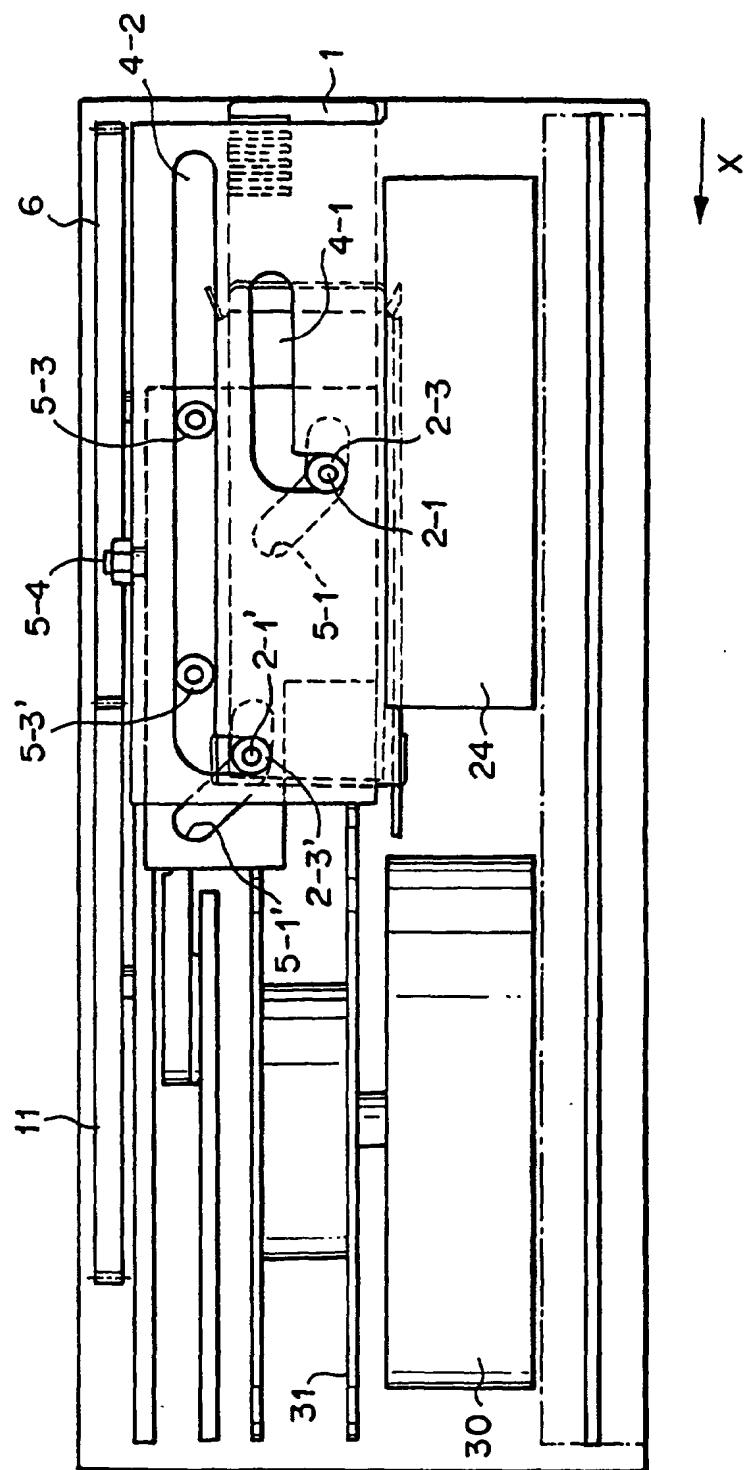
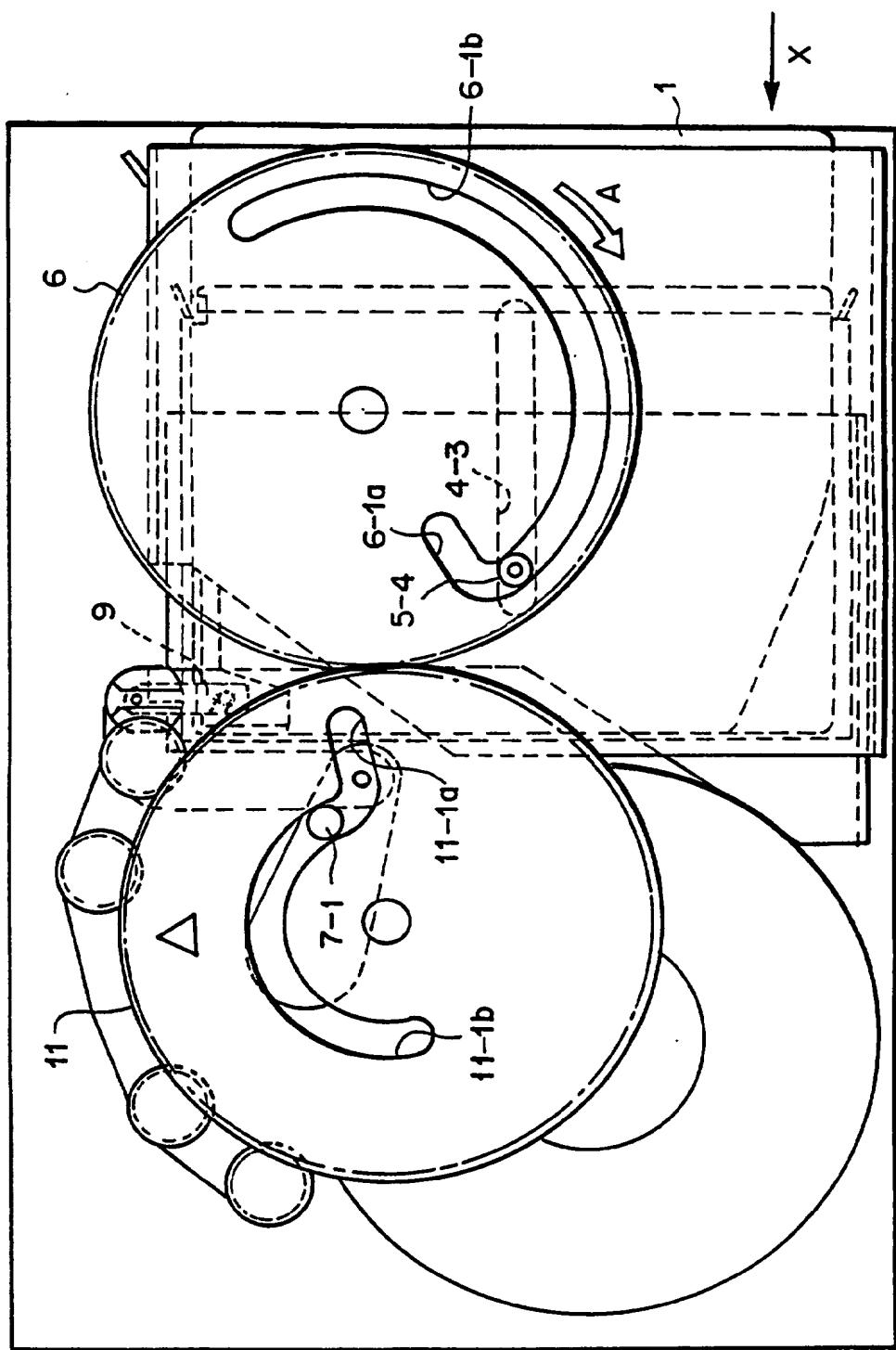
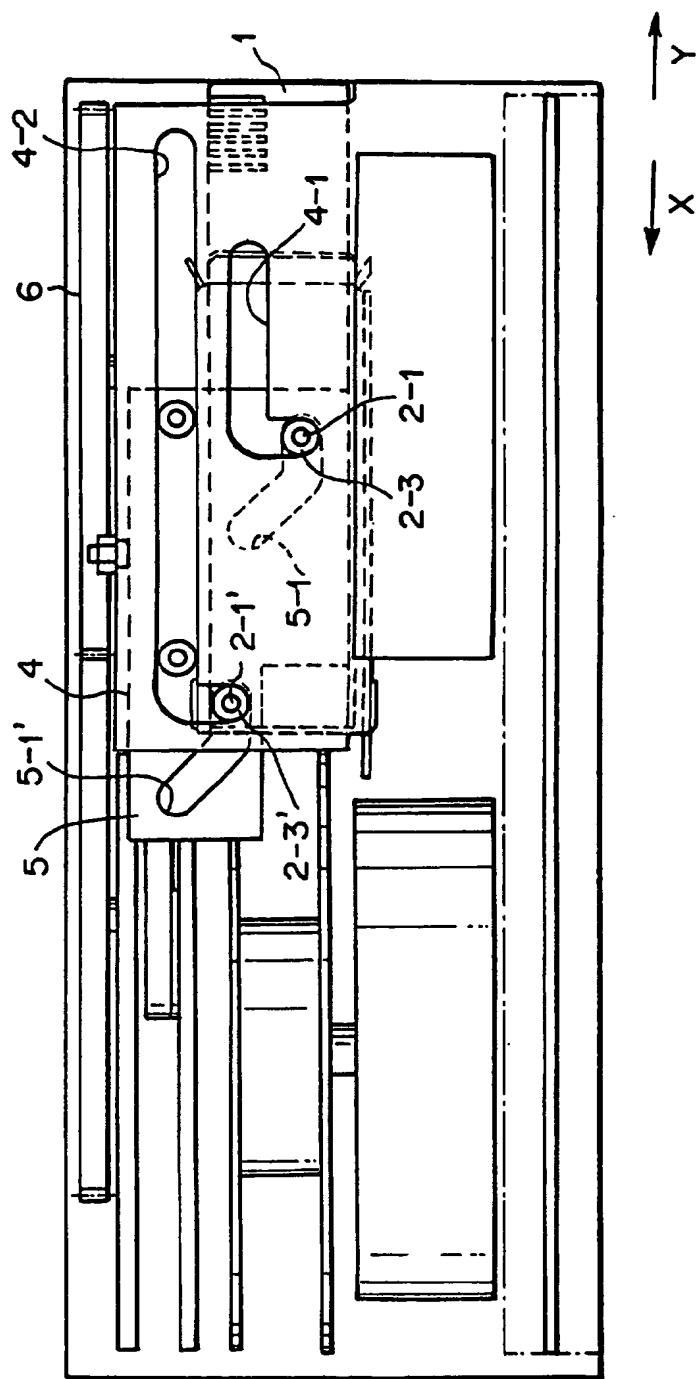


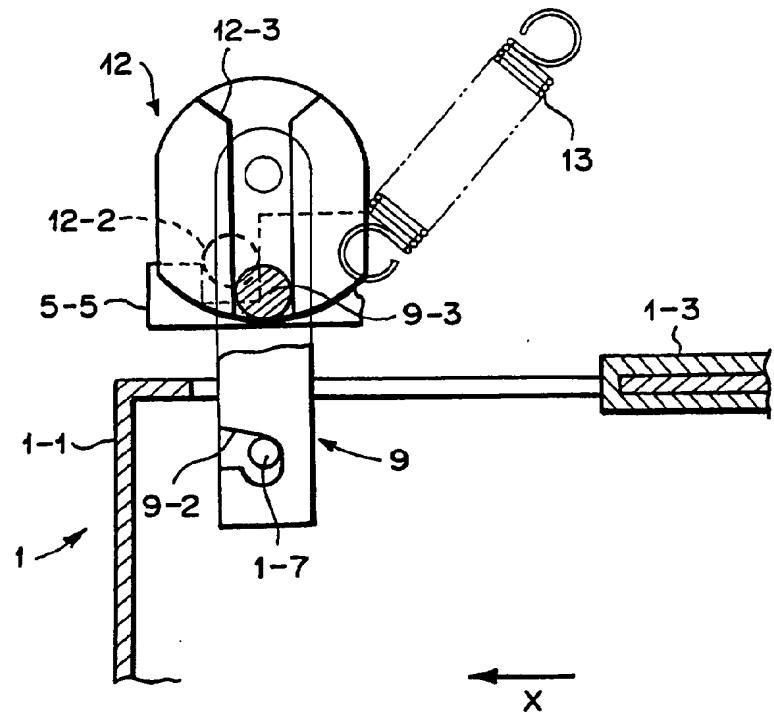
FIG. 16



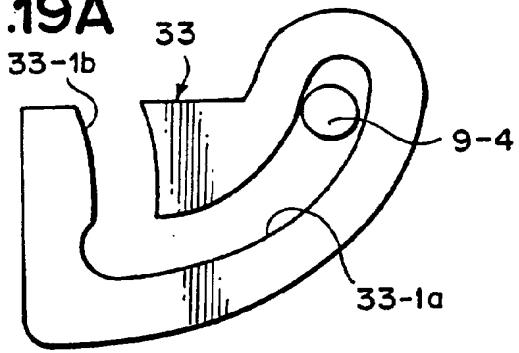
F I G. 17



**F I G.18**



**F I G.19A**



**F I G.19B**

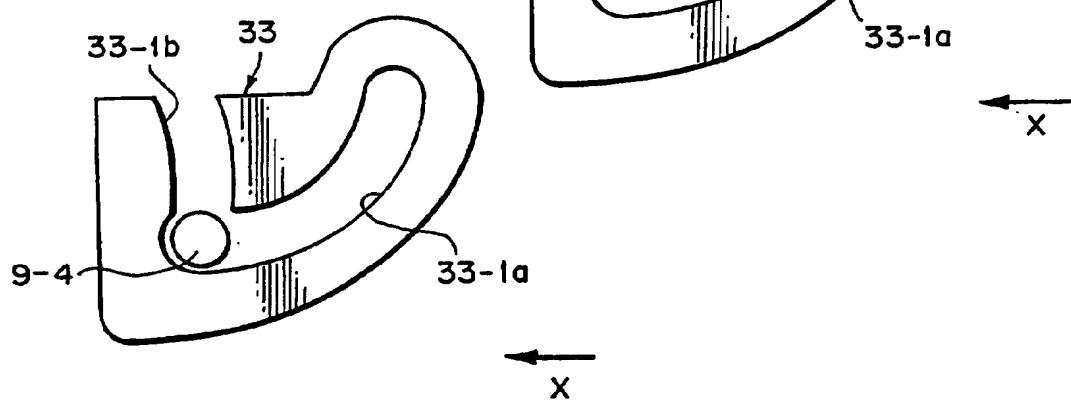


FIG. 20

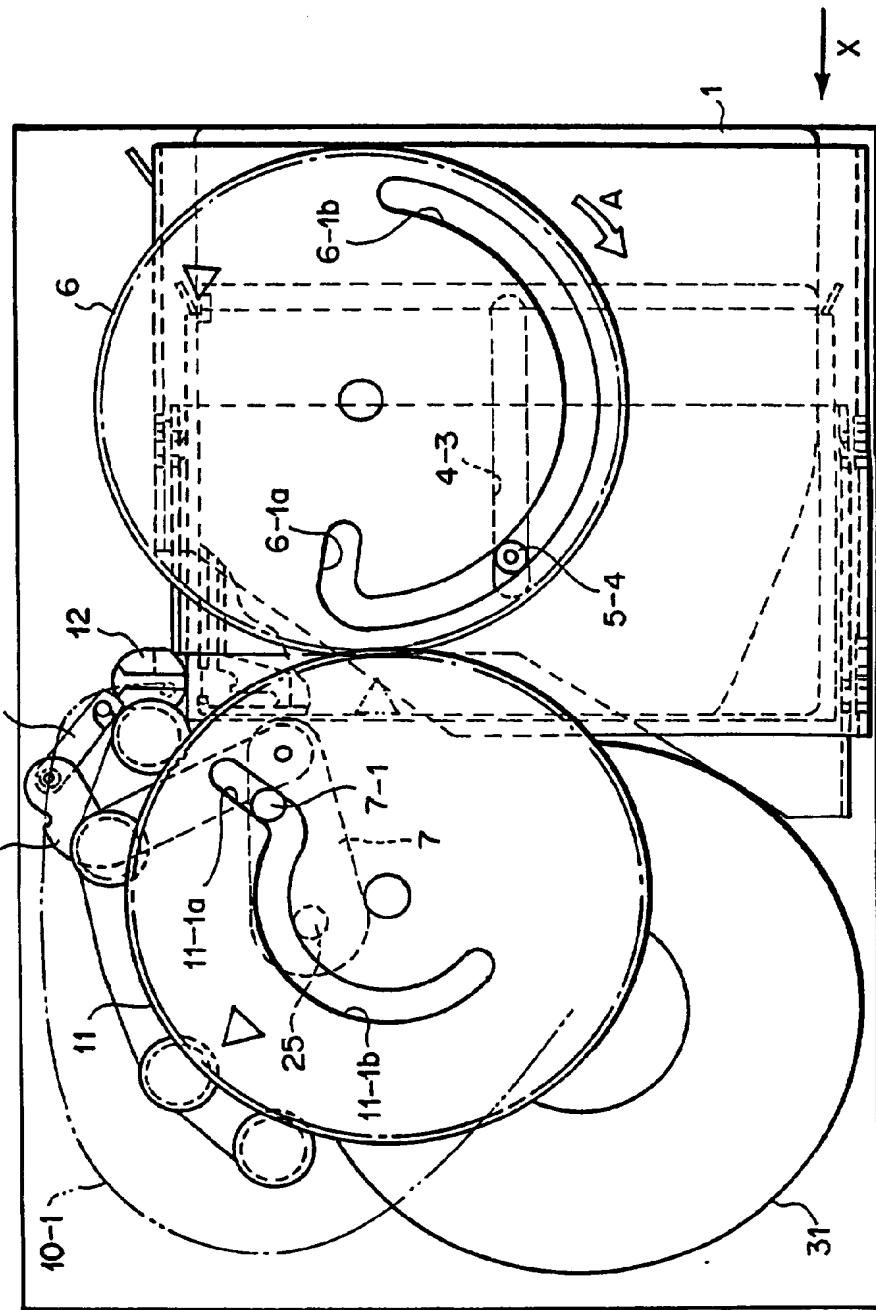
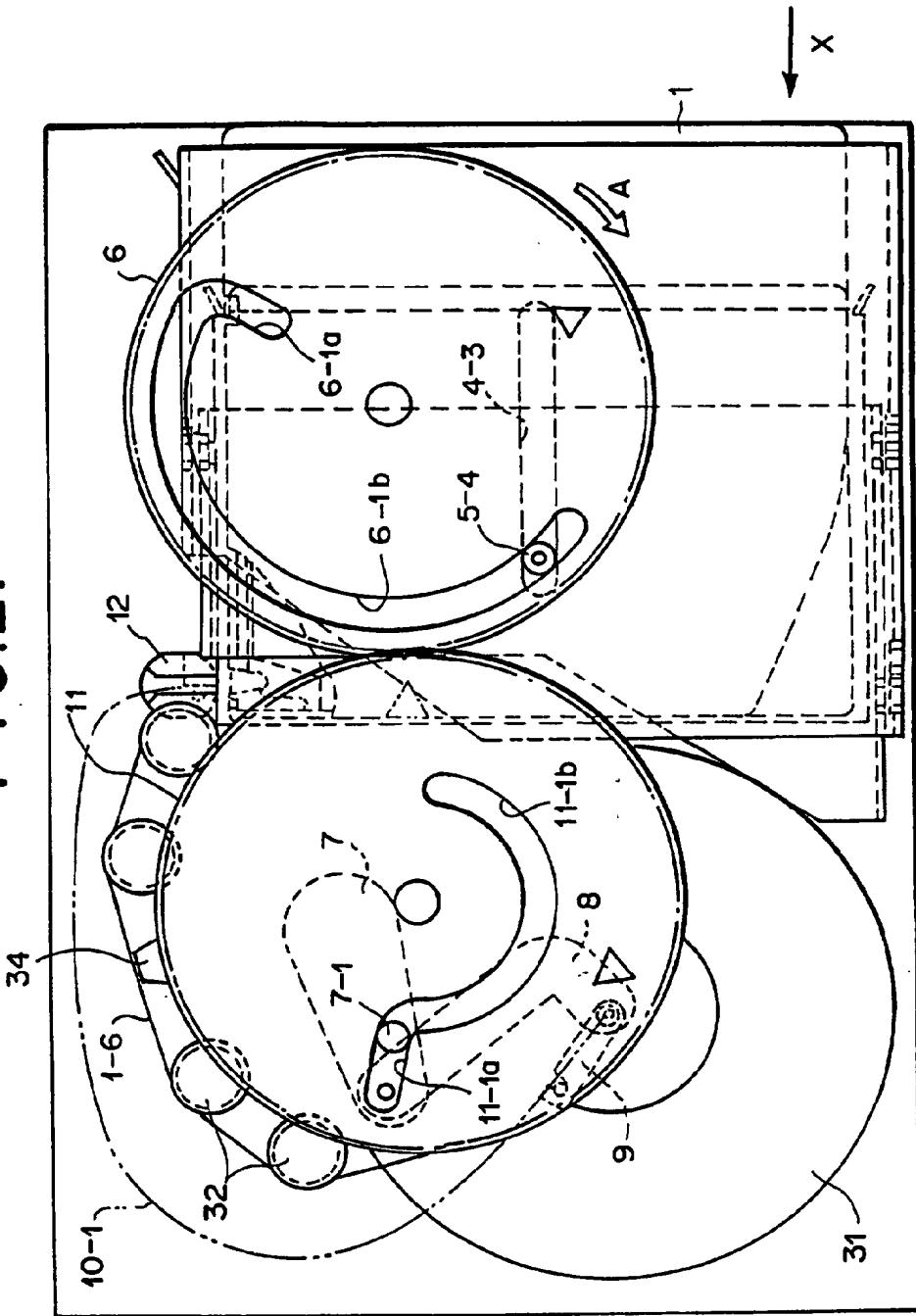
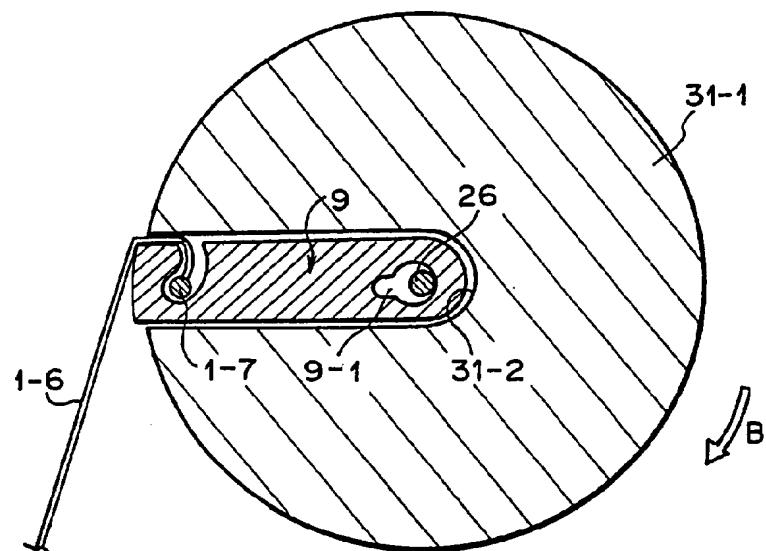


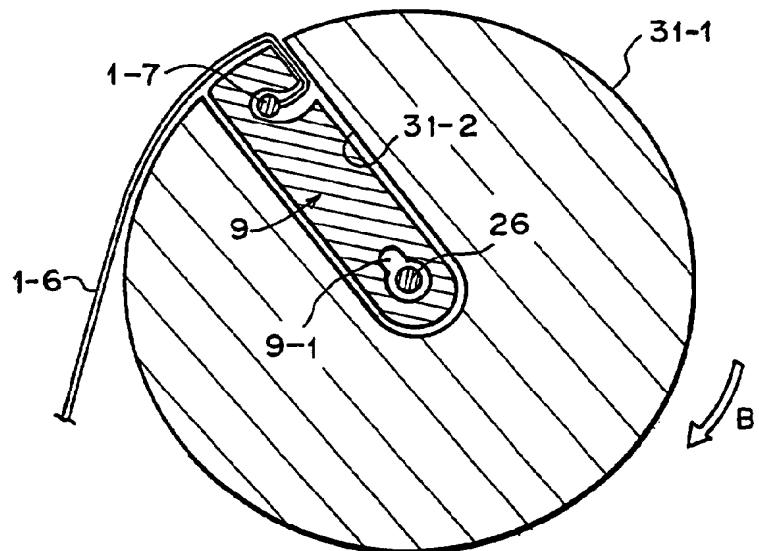
FIG. 21



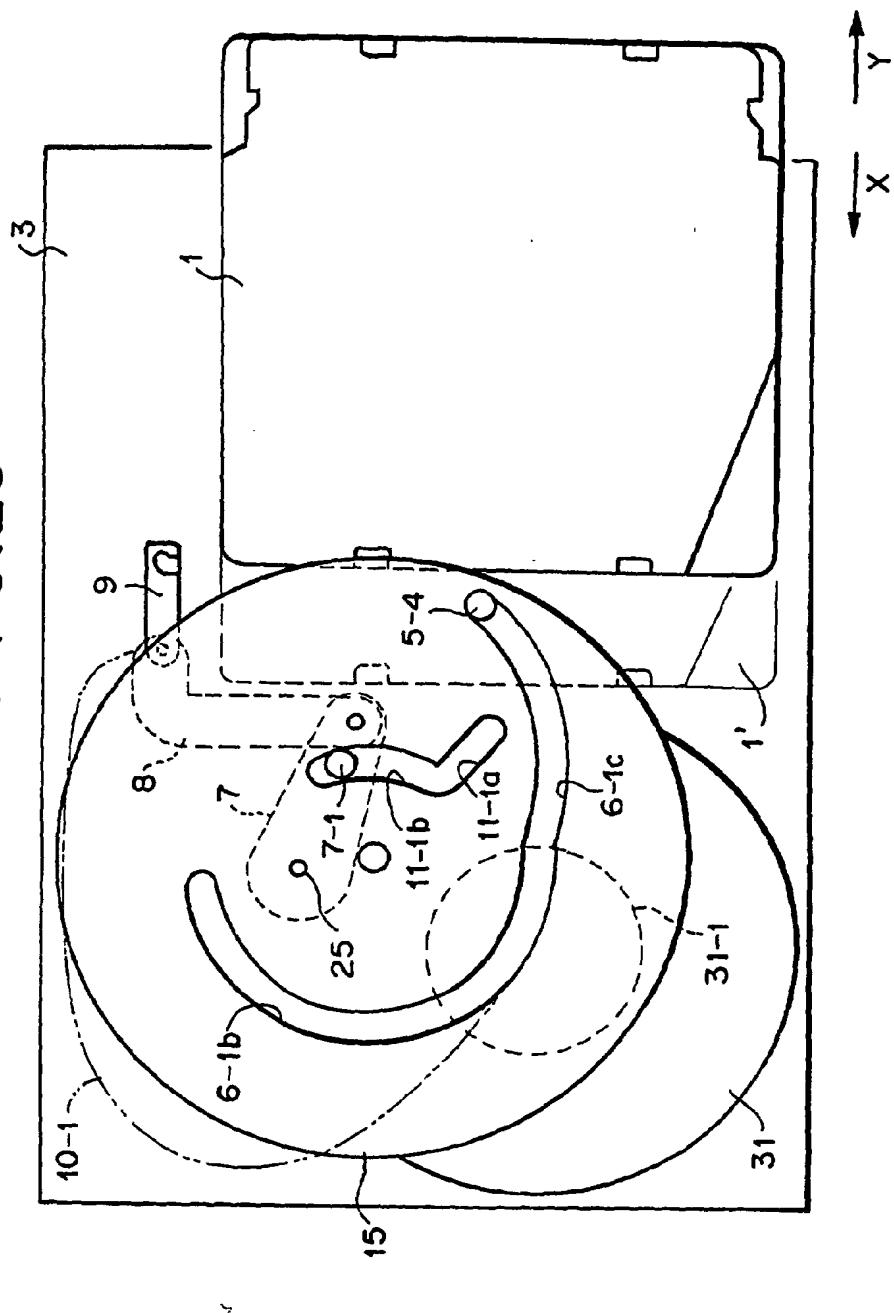
**F I G.22A**



**F I G.22B**



F I G. 23



**REFERENCES CITED IN THE DESCRIPTION**

*This list of references cited by the applicant is for the reader's convenience only. It does not form part of the European patent document. Even though great care has been taken in compiling the references, errors or omissions cannot be excluded and the EPO disclaims all liability in this regard.*

**Patent documents cited in the description**

- JP 8017111 A [0004]
- JP 9128855 A [0004]
- US 4991037 A [0006]
- WO 0030090 A [0007]
- WO 0030091 A [0008]
- WO 0030098 A [0009]